



What you need to know about residential furnaces, air conditioners and heat pumps if you're NOT an HVAC professional

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“This educational offering is recognized by the Minnesota Department of Labor and Industry as satisfying **1.5 hours** of credit toward **Building Officials and Residential Contractors** continuing education requirements.”

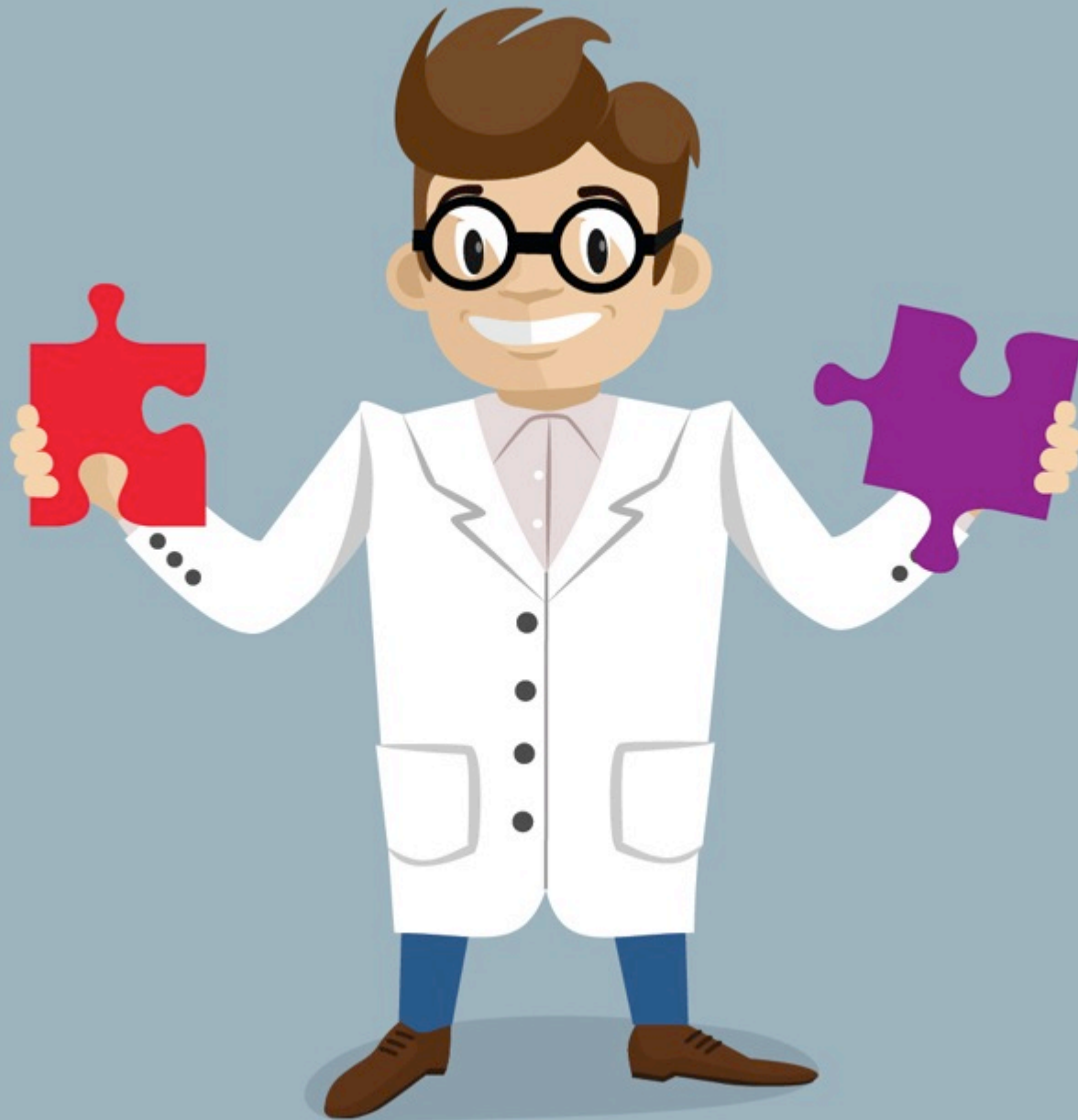
For additional continuing education approvals, please see your credit tracking card.

Test your knowledge!

1. Go to: **c3ping.com**
2. Enter Ping ID: **6160**
3. Enter a name
4. “Waiting for the next question...”

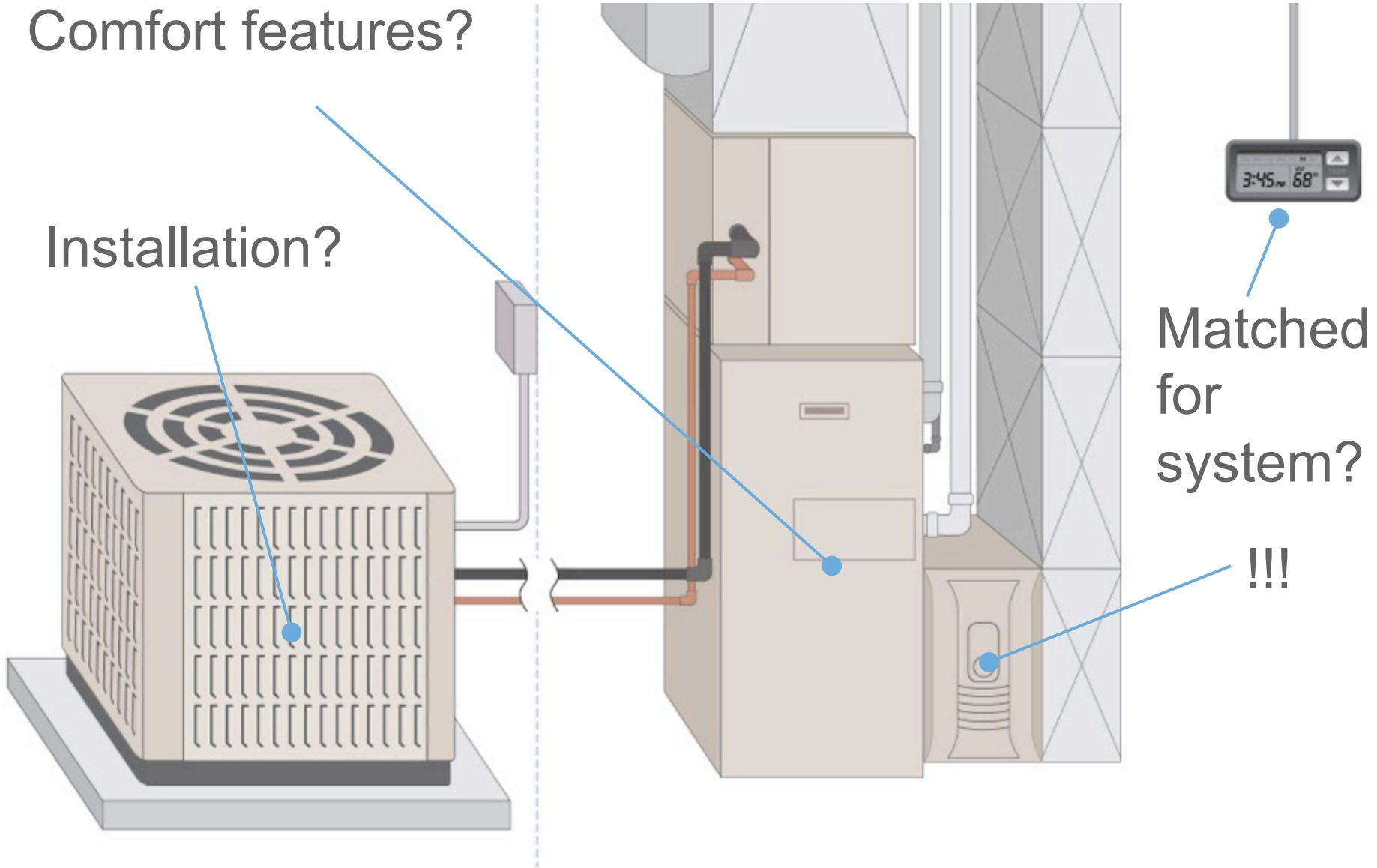
Who's here?

- A. Builder / remodeler
- B. Home Performance Consultant
- C. Weatherization provider
- D. HVAC contractor / distributor
- E. Other



Comfort features?

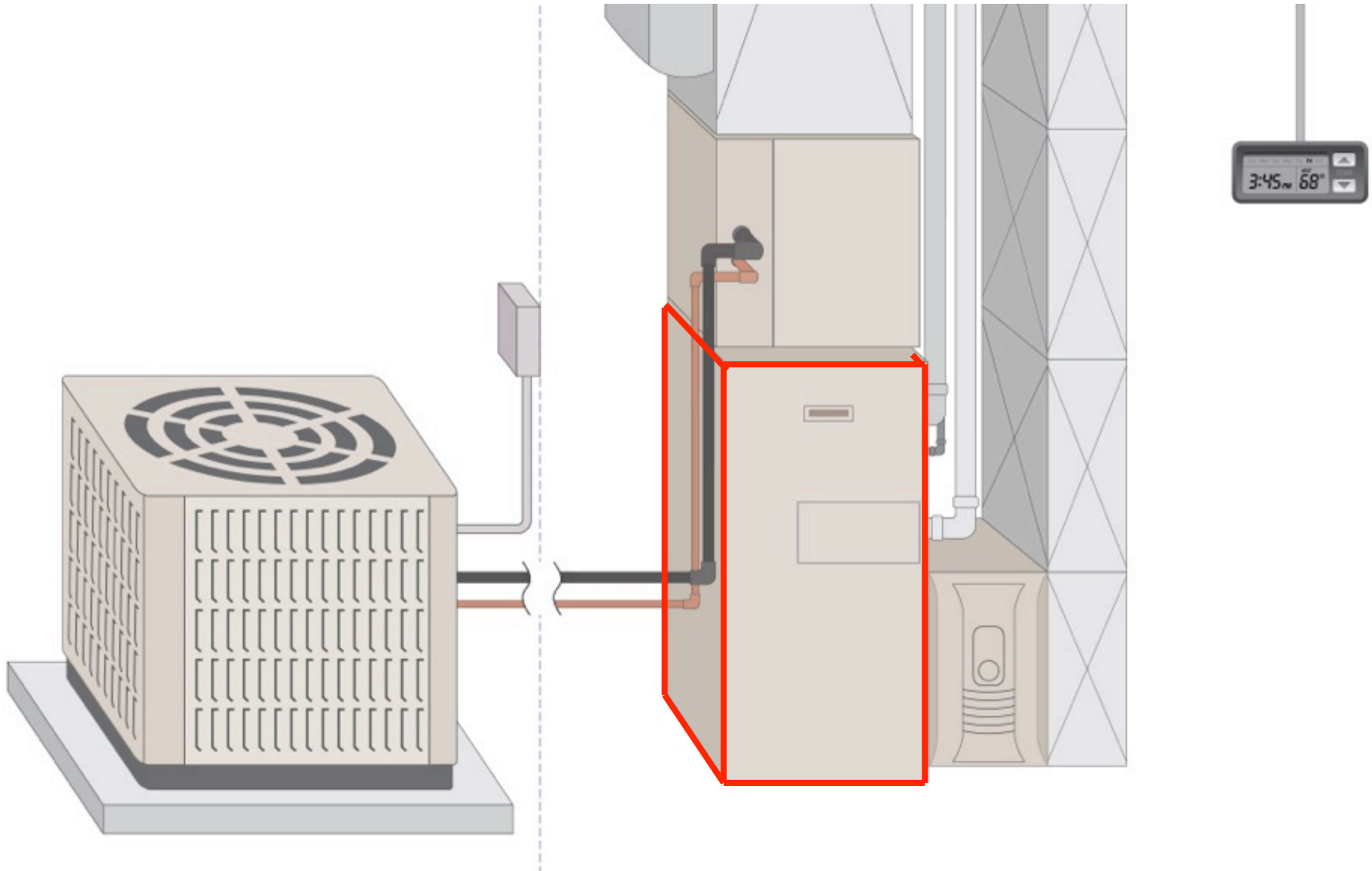
Installation?



Matched for system?

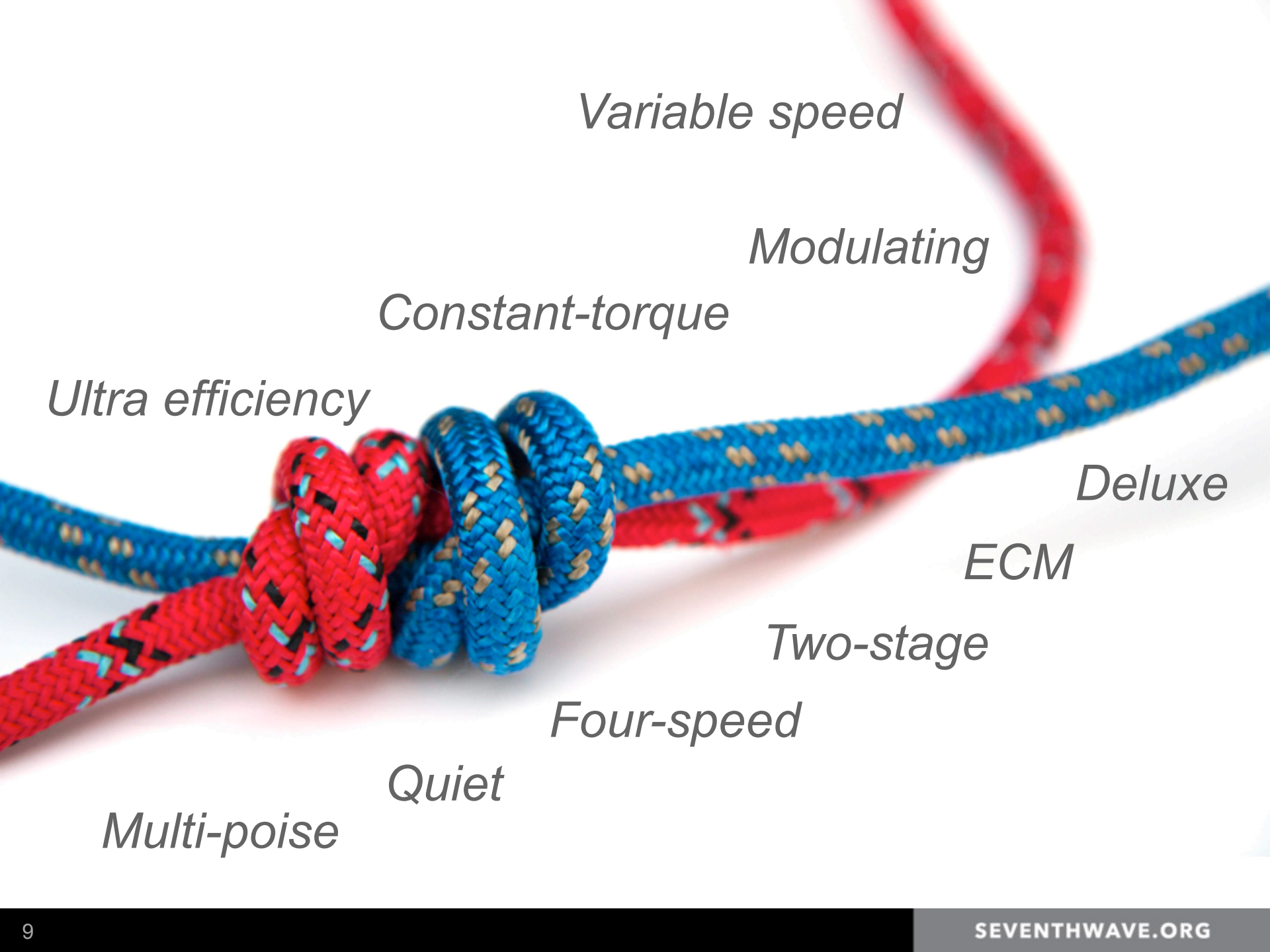
!!!

Furnace



The heating system in MY home is...

- A. ... a natural gas or propane furnace
- B. ... an oil furnace
- C. ... a hot-water or steam boiler
- D. ... something else



Variable speed

Modulating

Constant-torque

Ultra efficiency

Deluxe

ECM

Two-stage

Four-speed

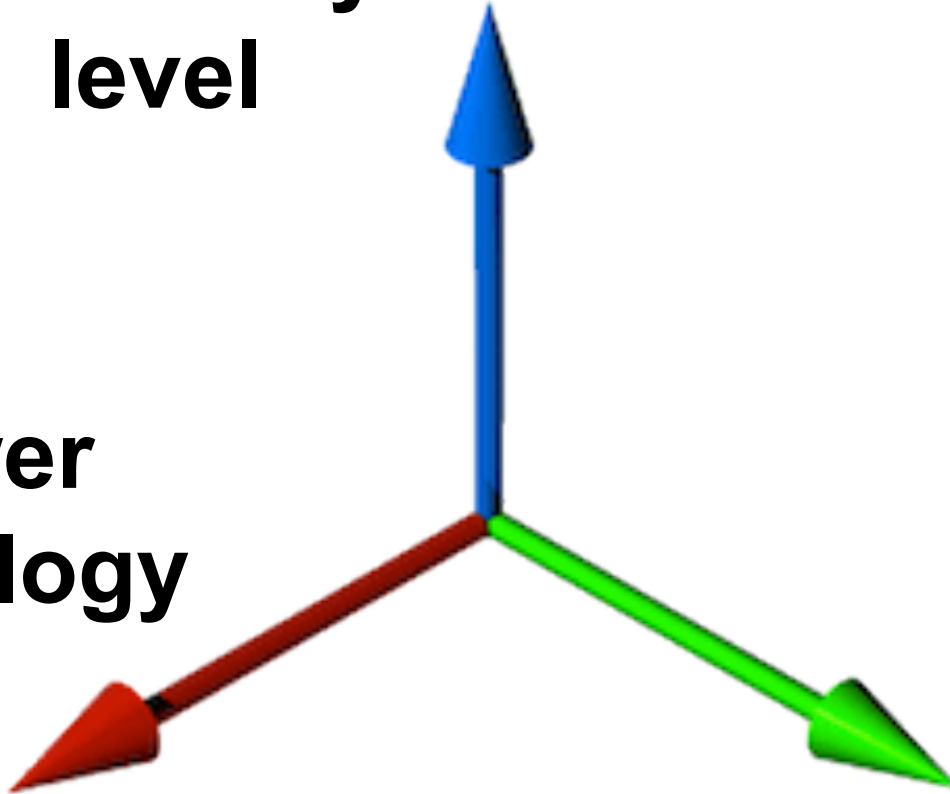
Quiet

Multi-poise

**Efficiency
level**

**Blower
technology**

**Firing
Stages**



Efficiency level

To condense...

*...or not to
condense?*



Non-condensing



The "gulf of corrosion"

Condensing

80%

85%

90%

95%

100%

Efficiency

MY furnace is...

- A. ...Non-condensing
- B. ...Condensing
- C. (I'm not sure)



The savings from upgrading from non-condensing to condensing are...

- A. 5%
- B. 10%
- C. 15%
- D. 20%
- E. 25%

The savings from upgrading from non-condensing to condensing are...

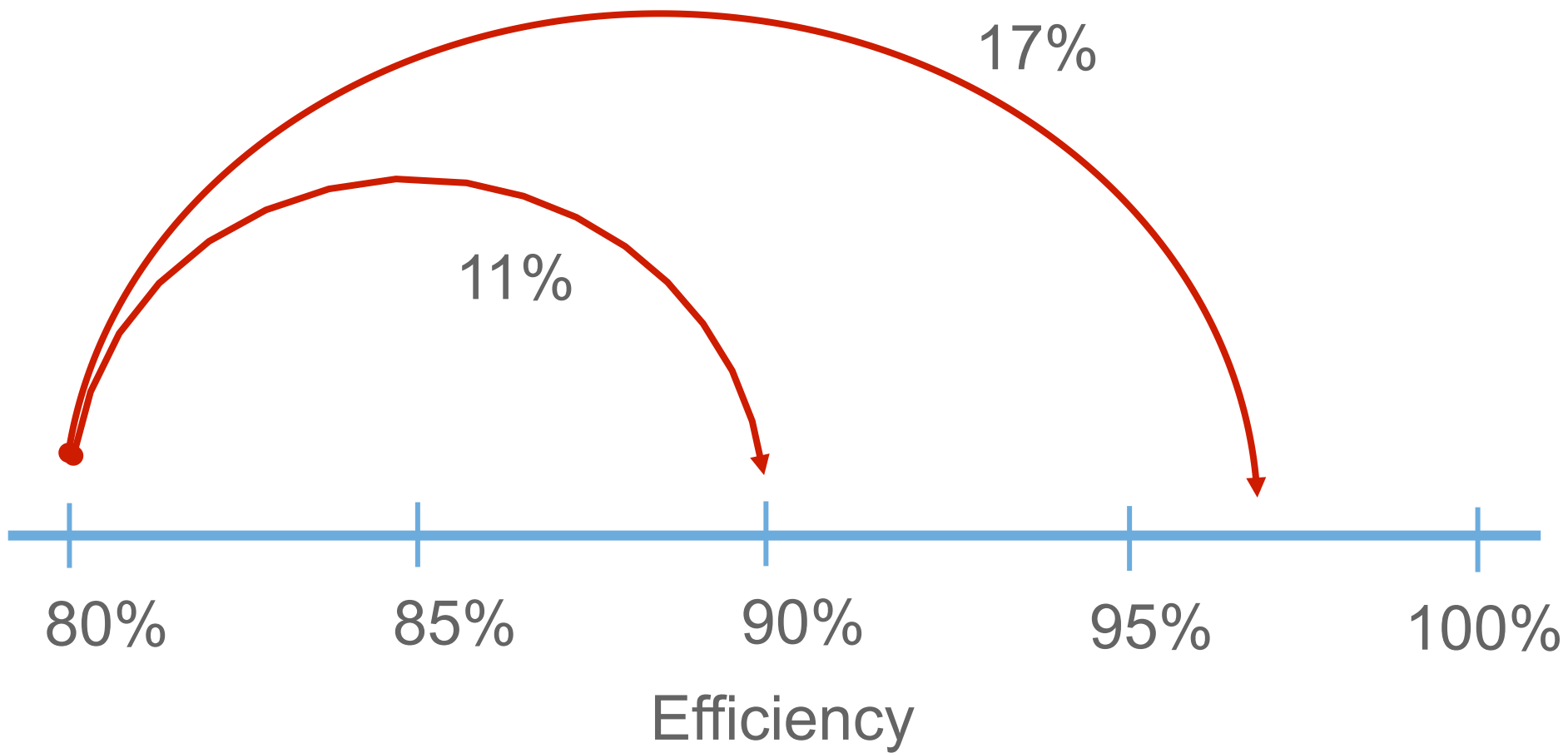
A. 5%

B. 10%

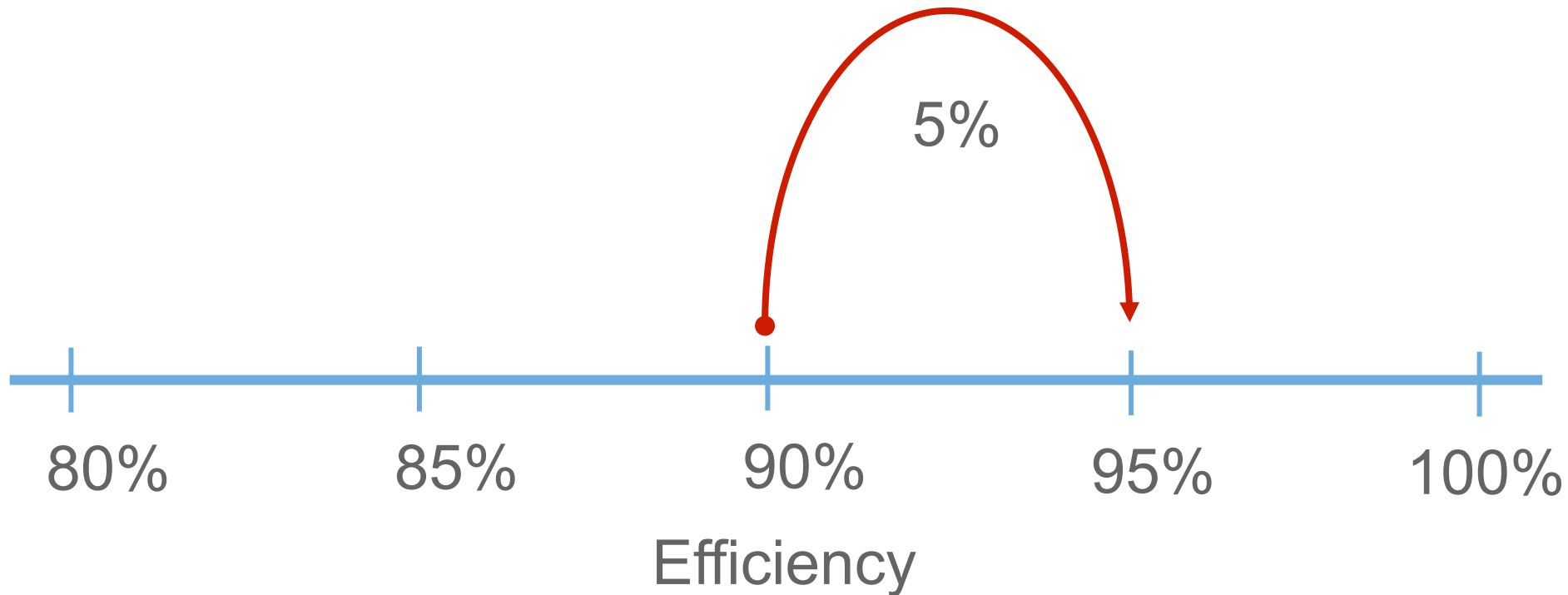
C. 15%

D. 20%

E. 25%



Upgrade to higher efficiency condensing?



Annual Fuel Utilization Efficiency AFUE

5 min,
10 min,
infinity;

75 min,
2.5 min,
5 min.

use $T_{F,OFF}(t_3) = T_{F,OFF}(t_2) + 0.1$,
use $T_{F,OFF}(t_3) = T_{F,OFF}(t_2) + 0.2$,
use $T_{F,OFF}(t_3) = T_{F,OFF}(t_2) + 0.1$.

Systems employing post purge with post-
purge times 9.1.1 and 9.5.1.2), the times t_3 , t_4 ,
follows.

$t_p + 1.5$ min,
 $t_p + 9.0$ min,
infinity.

$t_p + 3.75$ min,
 $t_p + 22.5$ min,
5 min.

as defined in 9.5.1.1 (furnaces) and
2 (boilers);

Temperature measured at time (t_3)
at shutdown from steady-state opera-
tion of the system burner, determined in
accordance with 9.5, in °F;

Temperature measured at time (t_4)
at shutdown from steady-state opera-
tion of the system burner, determined in
accordance with 9.5, in °F;

Minimum flue gas temperature for furnaces
and 45-minute reading for boilers,
determined in accordance with 9.5, in °F.

**Effective Flue Gas Temperature
Difference at Shutdown** Calculate the effective flue gas
temperature difference at shutdown of the system burner,
and defined as

$$(\psi_{S,O,X} - T_{F,OFF}(t_3)) e^{-(R_{ON} + R_{OFF})t_3}$$

defined in 11.2.9.6,
defined in 11.2.9.6,
defined in 11.2.9.6,
defined in 11.2.9.6.

**Minimum Flue Gas Temperature
Difference at Shutdown** Calculate the
temperature difference above room
temperature expressed in °F and defined as

$$T_{F,OFF}(t_3) - T_{RA}$$

where

$T_{F,OFF}(t_3)$ = value as defined in 11.2.9.6, and
 T_{RA} = value as defined in 11.2.5.

**11.2.9.9 Minimum Stack Gas Temperature
Difference above Room Temperature** Calculate the
minimum stack gas temperature difference above room
temperature, $\psi_{S,O,X}$ expressed in °F and defined as
follows.

For systems numbered 1-4 and systems 5-8 if $(S/F)(D_S^0) > D_P$,

$$\psi_{S,O,X} = \frac{(D_P)(\psi_{F,O,X})}{(S/F)(D_S^0)}$$

For systems numbered 5-8 for which $(S/F)(D_S^0) \leq D_P$,

$$\psi_{S,O,X} = \psi_{F,O,X}$$

where

D_P = off-cycle flue gas draft factor, which is equal
to:

- 1.0 for units with atmospheric burners, systems 1, 5, 9 (except direct vent and atmospheric burners with either electro-mechanical inlet dampers or flue dampers);
- 0.4 or D_P for units with power burners, systems 2, 3, 4, 6, 8, 10;
- 1.0 or D_P for direct vent units with atmospheric burners, system 9 (direct vent only);
- 1.0 or D_P for atmospheric burners with either electro-mechanical inlet dampers or flue dampers (system 1) and with units employing isolated combustion (system 9);

D_S^0 = off-cycle stack gas draft factor, without stack
damper, which is equal to:

- 1 for systems 1, 2, 5, and 6;
- 0.4 for system 3;
- 0.85 or $[(0.79 + D_P)/1.4]$ for systems 4 and 8.

where

D_P = the power burner draft factor as
determined by the optional pro-
cedure defined in 9.7,

S/F = value as defined in 11.2.4,

$\psi_{F,O,X}$ = value as defined in 11.2.9.8.

**11.2.9.10 Effective Stack Gas Temperature
Difference at Shutdown** Calculate the effective stack gas
temperature difference at shutdown, $\psi_{S,O,X}$ expressed in °F
and defined as follows.

For systems numbered 1-4 and 5-8 if $(S/F)(D_S^0) > D_P$,

$$\psi_{S,O,X} = \frac{(D_P)(\psi_{F,O,X})}{(S/F)(D_S^0)}$$

For systems 5-8 for which $(S/F)(D_S^0) \leq D_P$,

$$\psi_{S,O,X} = \psi_{F,O,X}$$

where

D_P = value as defined in 11.2.9.9,

$\psi_{F,O,X}$ = value as defined in 11.2.9.7,

S/F = value as defined in 11.2.4,

D_S^0 = value as defined in 11.2.9.9.

**11.2.9.11 Ratio of Average Burner On-Time
Per Cycle to On-Cycle Time Constant** Calculate the ratio
of average burner on-time per cycle to on-cycle time
constant, R_{ON} defined as

$$R_{ON} = t_{ON}/\tau_{ON}$$

where

for furnaces, t_{ON} = 3.87 min, the average burn-
er on-time per cycle, and
 τ_{ON} = value as defined in 11.2.9.4;

for boilers, t_{ON} = 9.68 min, the average burn-
er on-time per cycle, and
 τ_{ON} = value as defined in 11.2.9.4.

**11.2.9.12 Ratio of Average Burner Off-Time
Per Cycle to Off-Cycle Time Constant** Calculate the ratio
of average burner off-time per cycle to off-cycle time
constant, R_{OFF} defined as

$$R_{OFF} = t_{OFF}/\tau_{OFF}$$

where

for furnaces, t_{OFF} = 13.3 min, the average burn-
er off-time per cycle, and
 τ_{OFF} = value as defined in 11.2.9.6;

for boilers, t_{OFF} = 33.26 min, the average burn-
er off-time per cycle,
and
 τ_{OFF} = value as defined in 11.2.9.6.

**11.2.9.13 Start-up Burner Cycling Effect
Correction Factor** Calculate the burner start-up cycling
effect factor ($C_{T,ON}$) defined as

$$C_{T,ON} = \frac{1 - \frac{(\psi_{F,O,X})(e^{-R_{OFF}})}{(T_{F,SS} - T_{F,OFF}(t_3))}}{1 - \frac{(\theta_{F,O,X})(\psi_{F,O,X})(e^{-(R_{ON} + R_{OFF})})}{(T_{F,SS} - T_{F,OFF}(t_3))^2}}$$

where

$\theta_{F,O,X}$ = value as defined in 11.2.9.5,

$\psi_{F,O,X}$ = value as defined in 11.2.9.7,

R_{OFF} = value as defined in 11.2.9.12,

R_{ON} = value as defined in 11.2.9.11,

$T_{F,SS}$ = value as defined in 11.2.5,

$T_{F,OFF}(t_3)$ = value as defined in 11.2.9.6.

**11.2.9.14 Shutdown Burner Cycling Effect
Correction Factor** Calculate the burner shutdown cycling
effect factor ($C_{T,OFF}$) defined as

$$C_{T,OFF} = \frac{C_{IID} \left[1 - \frac{(\theta_{F,O,X})(e^{-R_{ON}})}{(T_{F,SS} - T_{F,OFF}(t_3))} \right]}{1 - \frac{(\theta_{F,O,X})(\psi_{F,O,X})(e^{-(R_{ON} + R_{OFF})})}{(T_{F,SS} - T_{F,OFF}(t_3))^2}}$$

where

$\psi_{F,O,X}$ = value as defined in 11.2.9.7,

$\theta_{F,O,X}$ = value as defined in 11.2.9.5,

R_{OFF} = value as defined in 11.2.9.12,

R_{ON} = value as defined in 11.2.9.11,

$T_{F,SS}$ = value as defined in 11.2.5,

$T_{F,OFF}(t_3)$ = value as defined in 11.2.9.6,

C_{IID} = 1, for units with continuously burning
pilot lights; 0.90 for units with intermit-
tent or interrupted ignition devices.

**11.2.9.15 Effective Flue Gas Temperature
Difference at Burner Shutdown, Corrected for Burner
Cycling Effect** Calculate the effective flue gas temperature
difference at burner shutdown, corrected for burner cycling
effect, $\psi_{F,O}$

For systems numbered 1-8,

$$\psi_{F,O} = (\psi_{F,O,X})(C_{T,OFF})$$

For systems 9-10,

$$\psi_{F,O} = (C_{T,OFF})(C_S^1)(\psi_{F,O,X})$$

where

$\psi_{F,O,X}$ = value as defined in 11.2.9.7,

$C_{T,OFF}$ = value as defined in 11.2.9.14.

The correction factor, which corrects the effect of outdoor
air passing through the heat exchanger during the off-
period, C_S^1 , is defined as

$$C_S^1 = 1.22$$

**11.2.9.16 Effective Minimum Flue Gas Temper-
ature Difference above Room Temperature, Corrected
for Burner Cycling Effect** Calculate the difference above
room temperature, $\psi_{F,w}$, corrected for burner cycling
effect, expressed in °F and defined as follows.

For systems numbered 1-8,

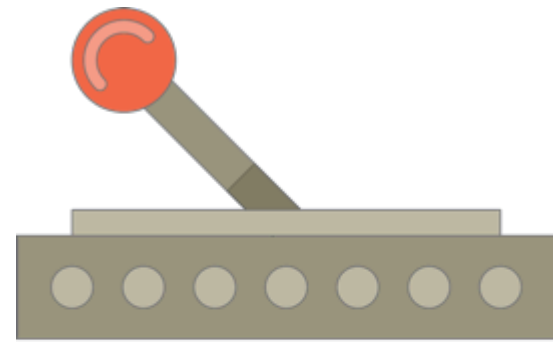
$$\psi_{F,w} = \psi_{F,w,X}$$

For systems numbered 9-10,

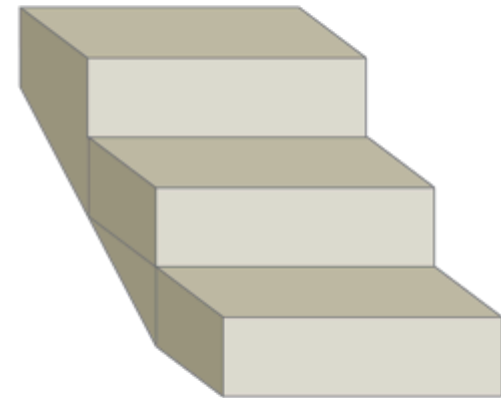
$$\psi_{F,w} = (C_S^1)(\psi_{F,w,X})$$

Firing stages

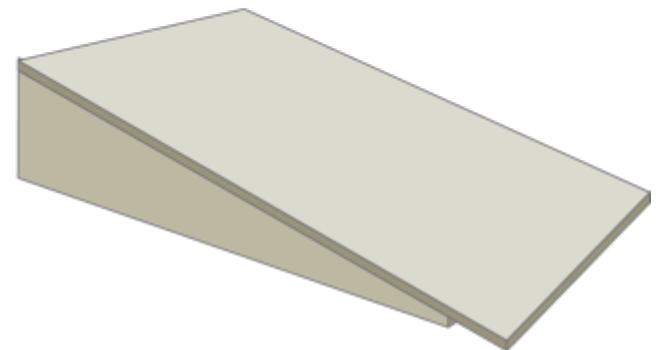
Single-stage



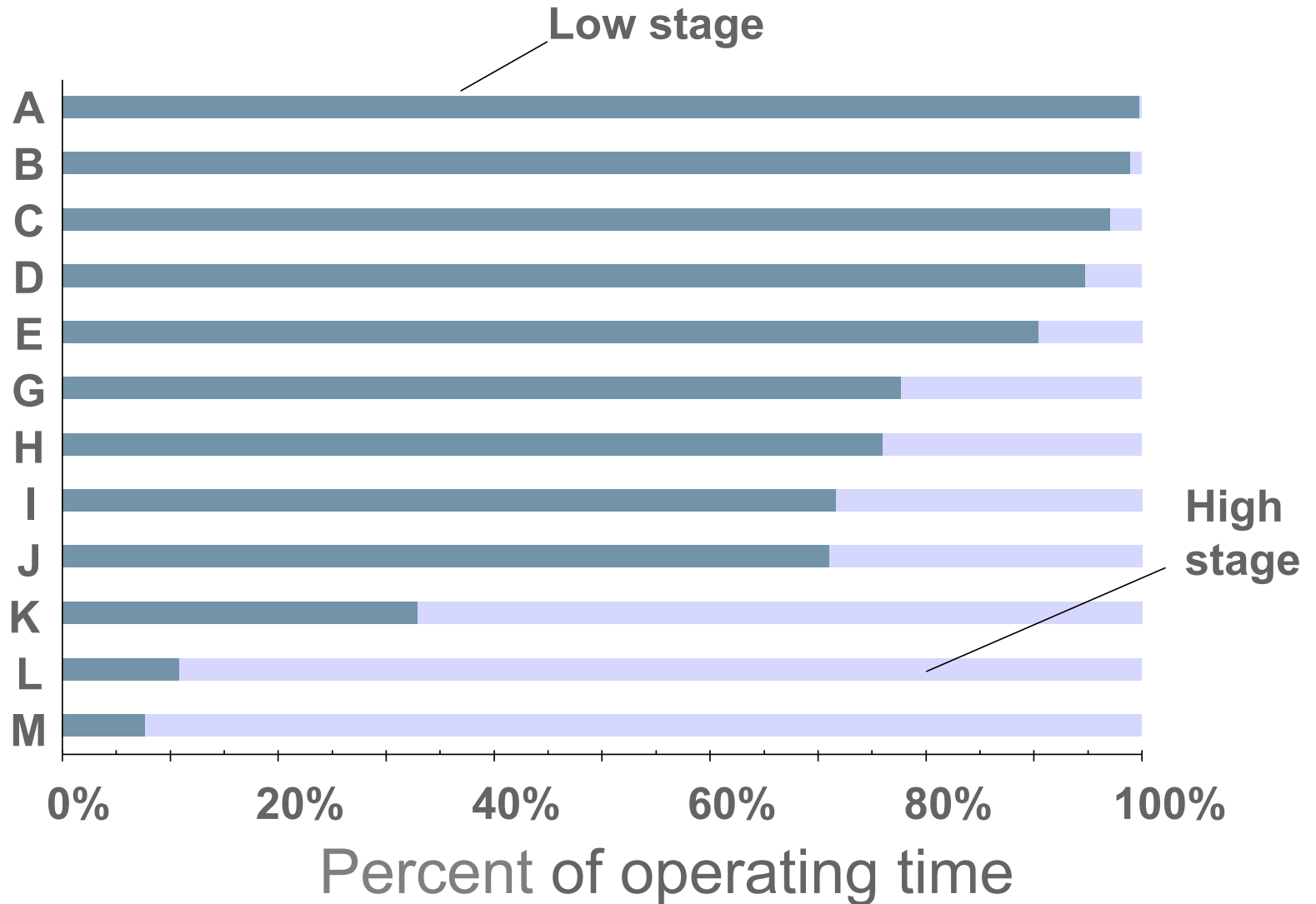
Multi-stage



Modulating



Two-stage operation (12 monitored furnaces)



Typical modulating furnace

Percent of full output

100

80

60

40

0

20

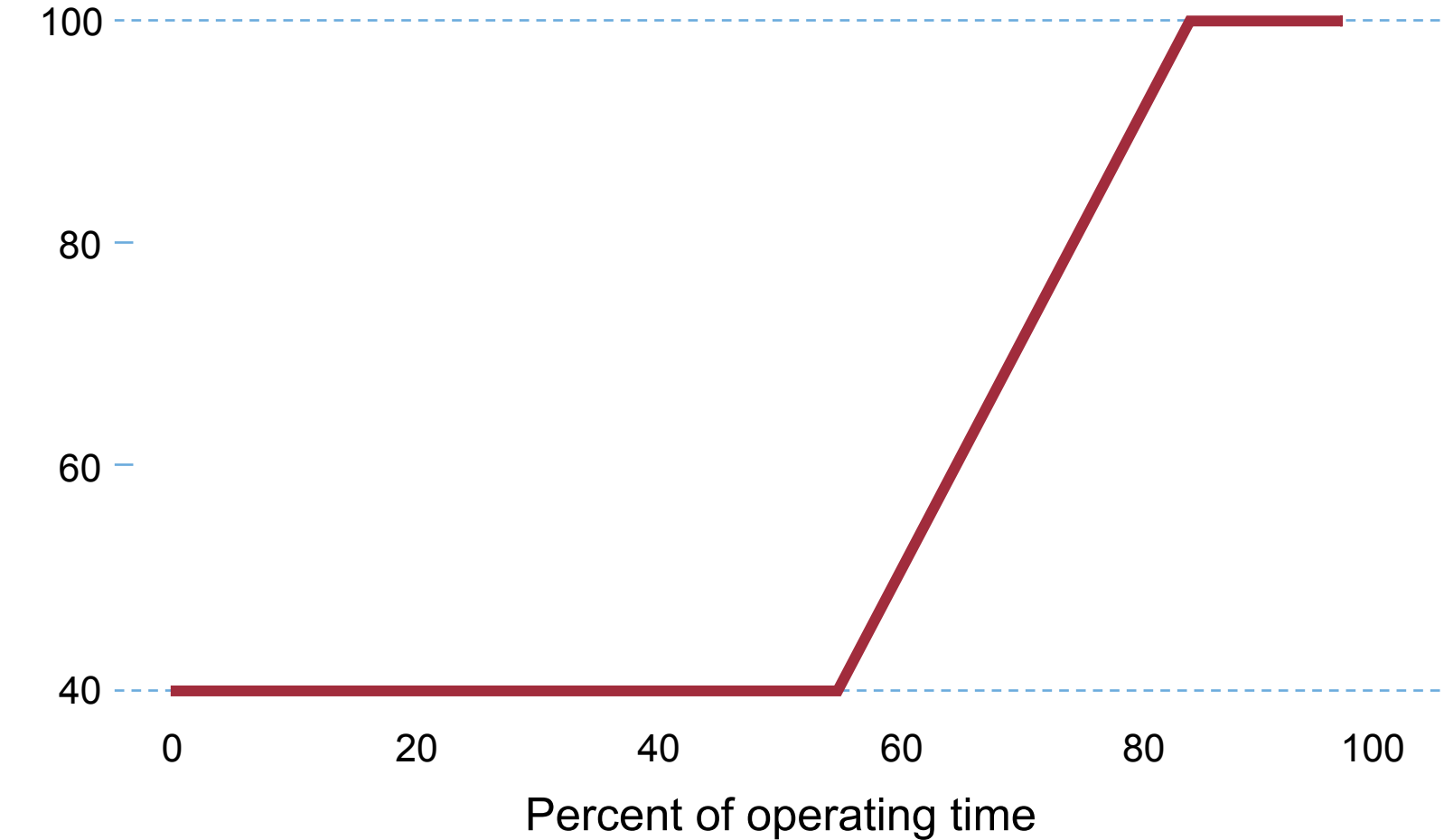
40

60

80

100

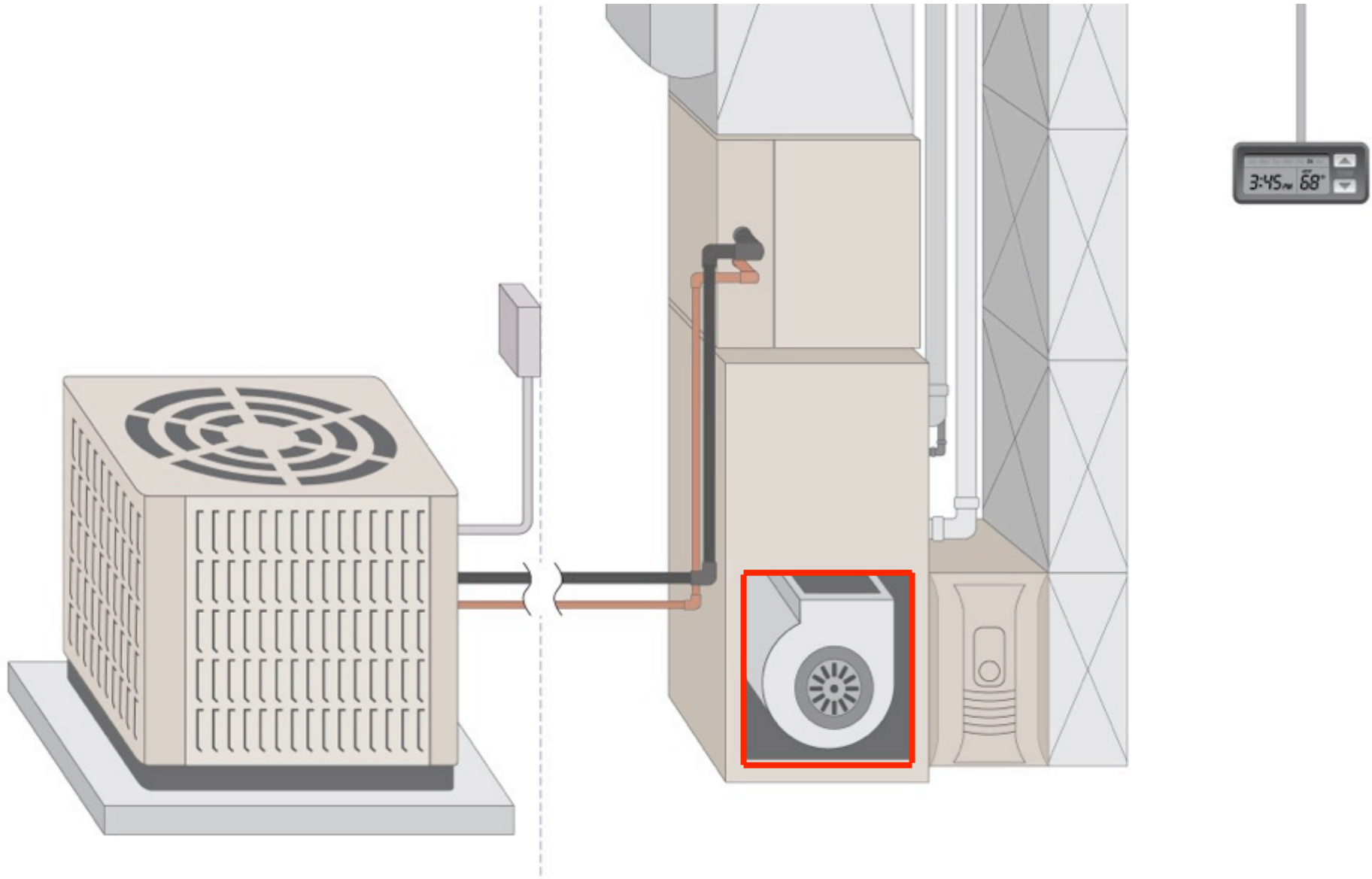
Percent of operating time



MY furnace is...

- A. ...Single-stage
- B. ...Multi-stage
- C. ...Modulating
- D. (I'm not sure)

Blower technology



PSC (Permanent Split-capacitor)



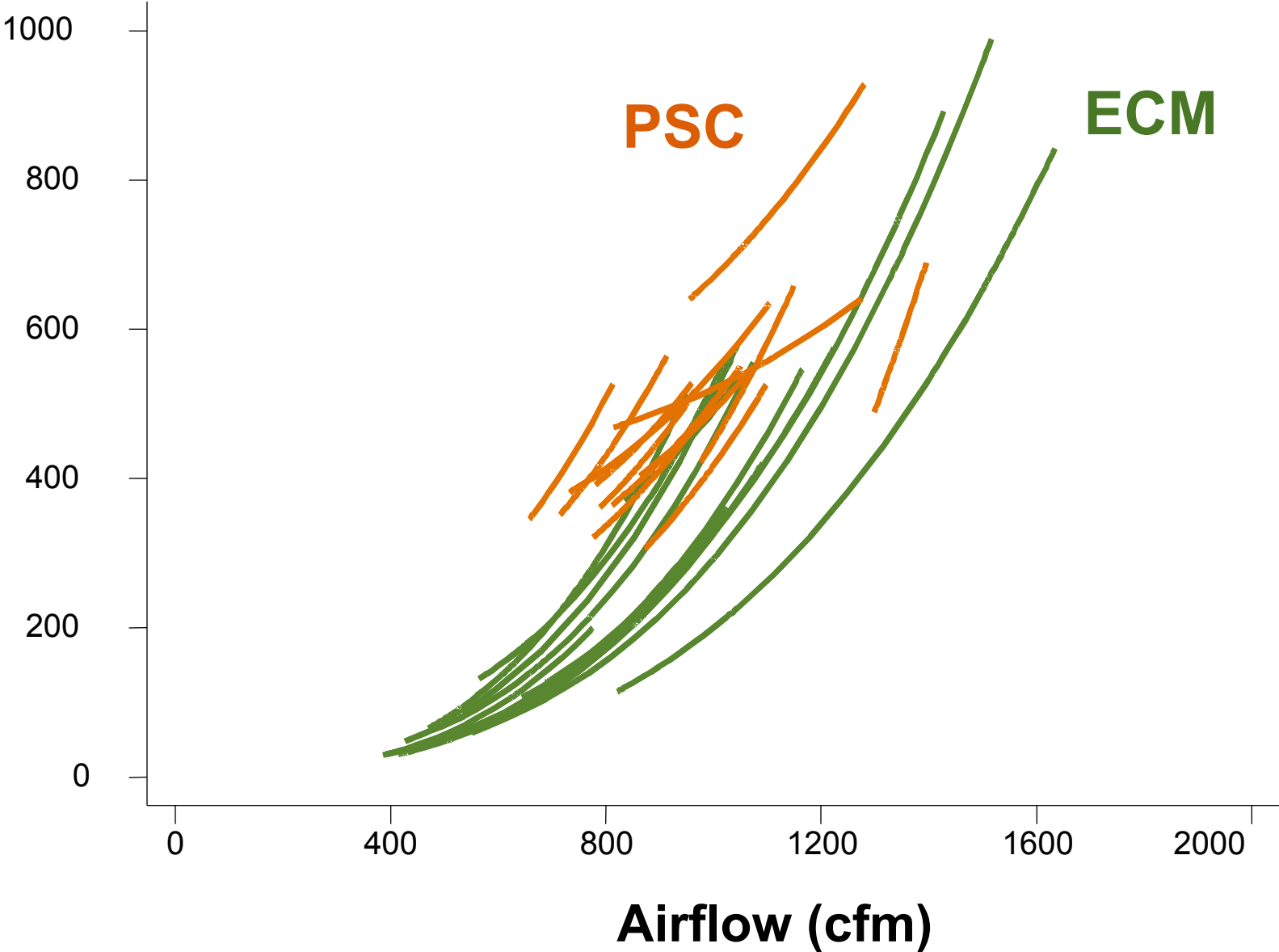
AKA
“Multi-speed”

ECM (Electronically commutated motor)

AKA
“variable speed”



Electricity consumption (watts)



Yearly ECM electricity savings

Fan "ON" use

before

after



= \$60

= \$450

= \$0

(@ 13 cents/kWh)

X-13 (trade name)



AKA
“constant torque”

MY furnace has...

- A. ...a PSC blower
- B. ...an ECM blower
- C. ...an X-13 blower
- D. (I'm not sure)

Performance (efficiency & comfort)



Non-condensing Single-stage PSC

Non-Condensing Multi-stage PSC

Condensing Single-stage PSC

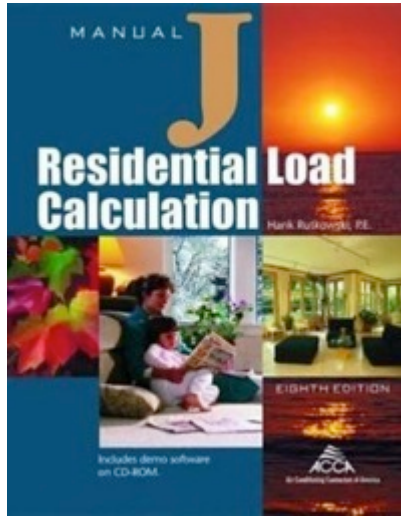
Condensing Multi-stage X-13

Condensing Multi-stage ECM

Condensing Modulating ECM

Price

Furnace sizing



40,000 Btuh



60,000 Btuh



80,000 Btuh



100,000 Btuh



Minnesota energy code:

“Oversizing of heating equipment shall not exceed ____ percent of the calculated load requirement”

- A. ...5
- B. ...10
- C. ...20
- D. ...40

Minnesota energy code:

“Oversizing of heating equipment shall not exceed ____ percent of the calculated load requirement”

- A. ...5
- B. ...10
- C. ...20
- D. ...40

What percent of Minnesota furnaces exceed the code limit for oversizing?

- A. 15%
- B. 30%
- C. 60%
- D. 95%

What percent of Minnesota furnaces exceed the code limit for oversizing?

A. 15%

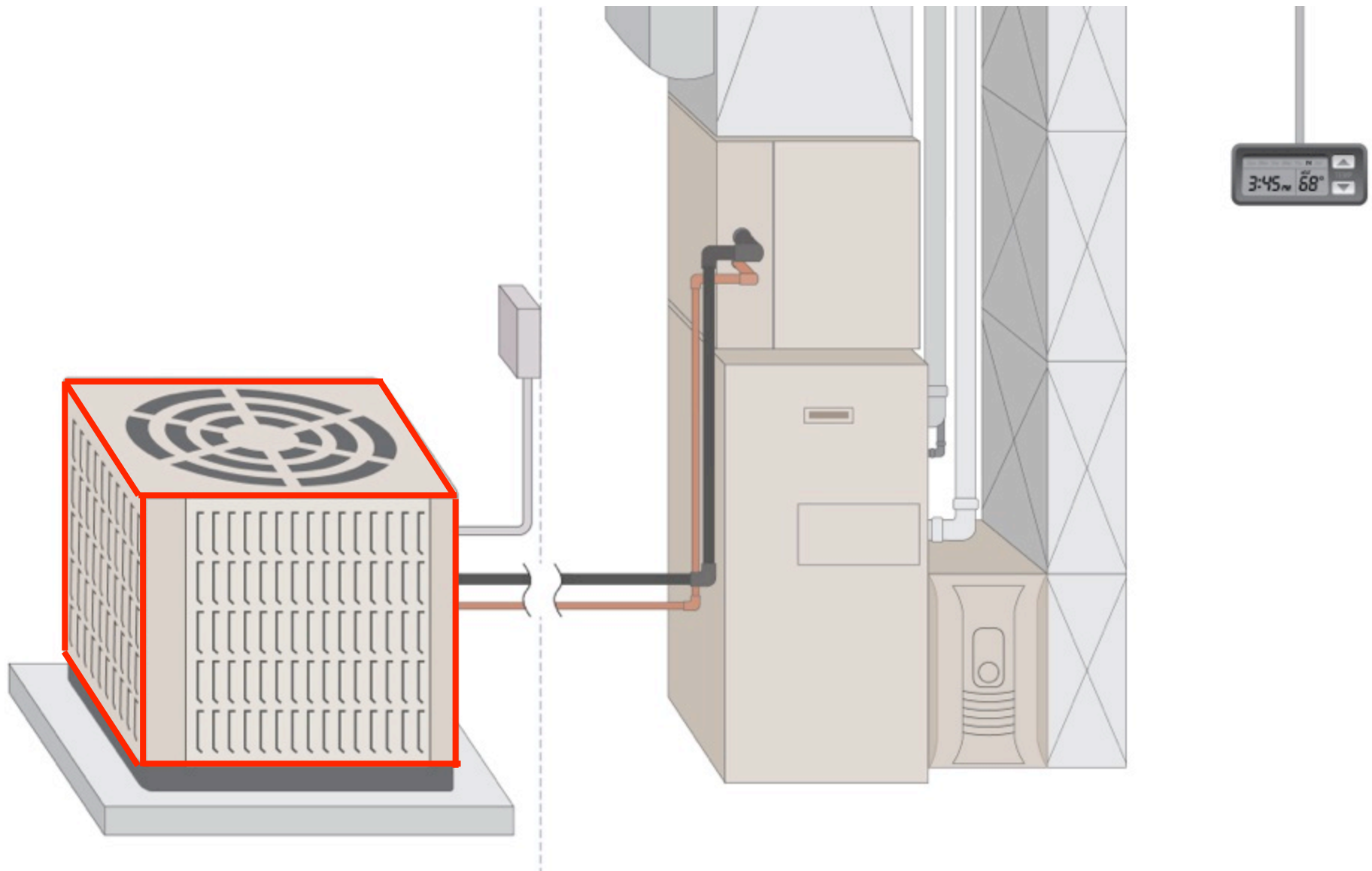
B. 30%

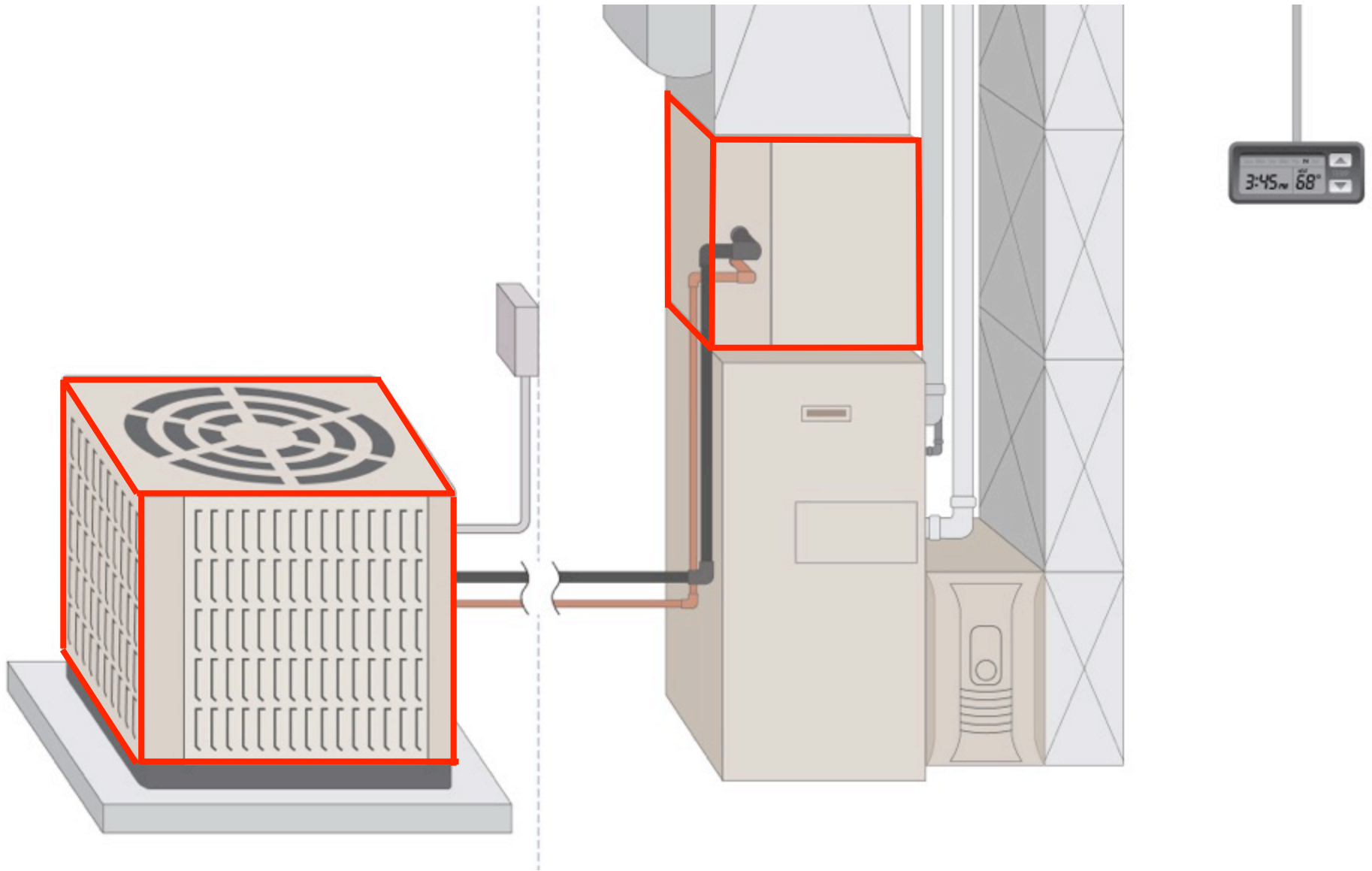
C. 60%

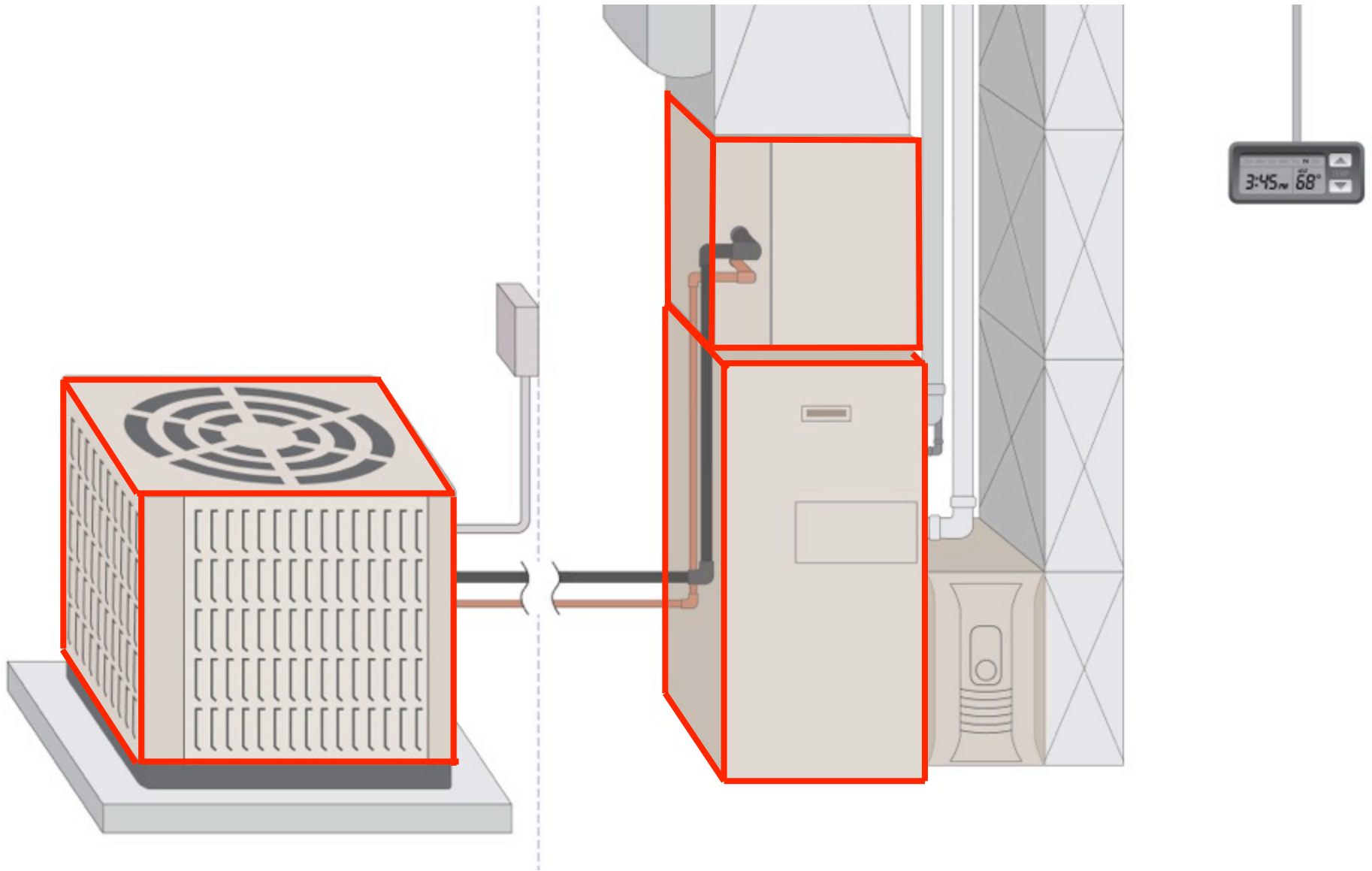
D. 95%

Questions about furnaces?

Air conditioner







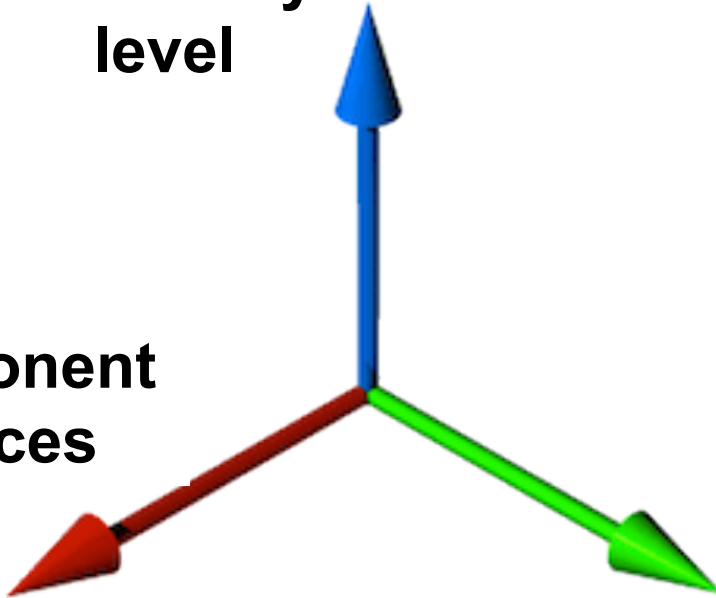


**Efficiency
level**

Installation?

**Component
choices**

**Cooling
Stages**



Seasonal Energy Efficiency Ratio SEER

How many hours per year does the average central air conditioner in Minneapolis run?

- A. ...120 hours
- B. ...240 hours
- C. ...325 hours
- D. ...450 hours
- E. ...630 hours

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A. ...120 hours

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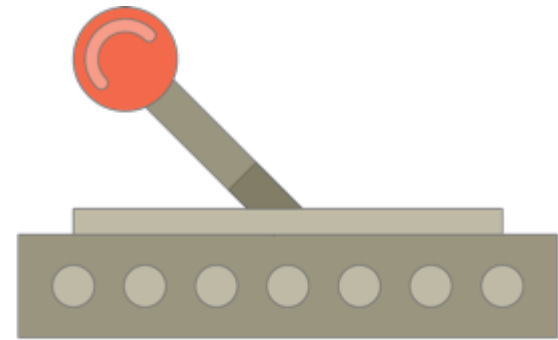
SEER upgrade savings

| SEER | Savings (vs SEER 13) |
|------|----------------------|
| 13 | \$0 |
| 14 | \$20 |
| 15 | \$35 |
| 16 | \$50 |
| 17 | \$65 |
| 18 | \$75 |

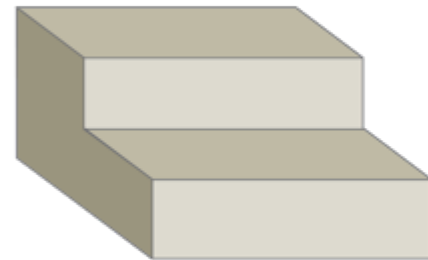
(2.25 tons capacity, 325 annual hours, 13 cents/kWh)

Cooling stages

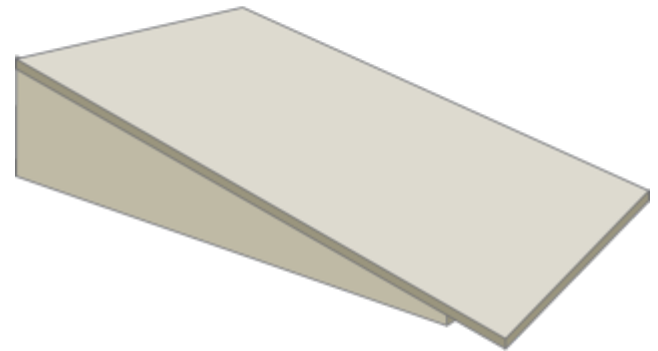
Single-stage



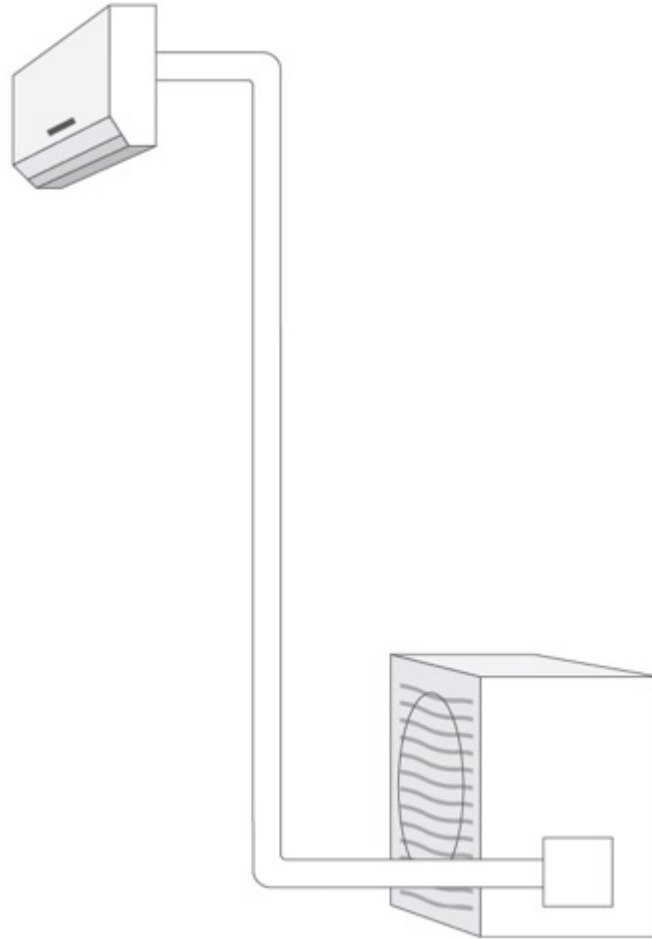
Two-stage



Modulating



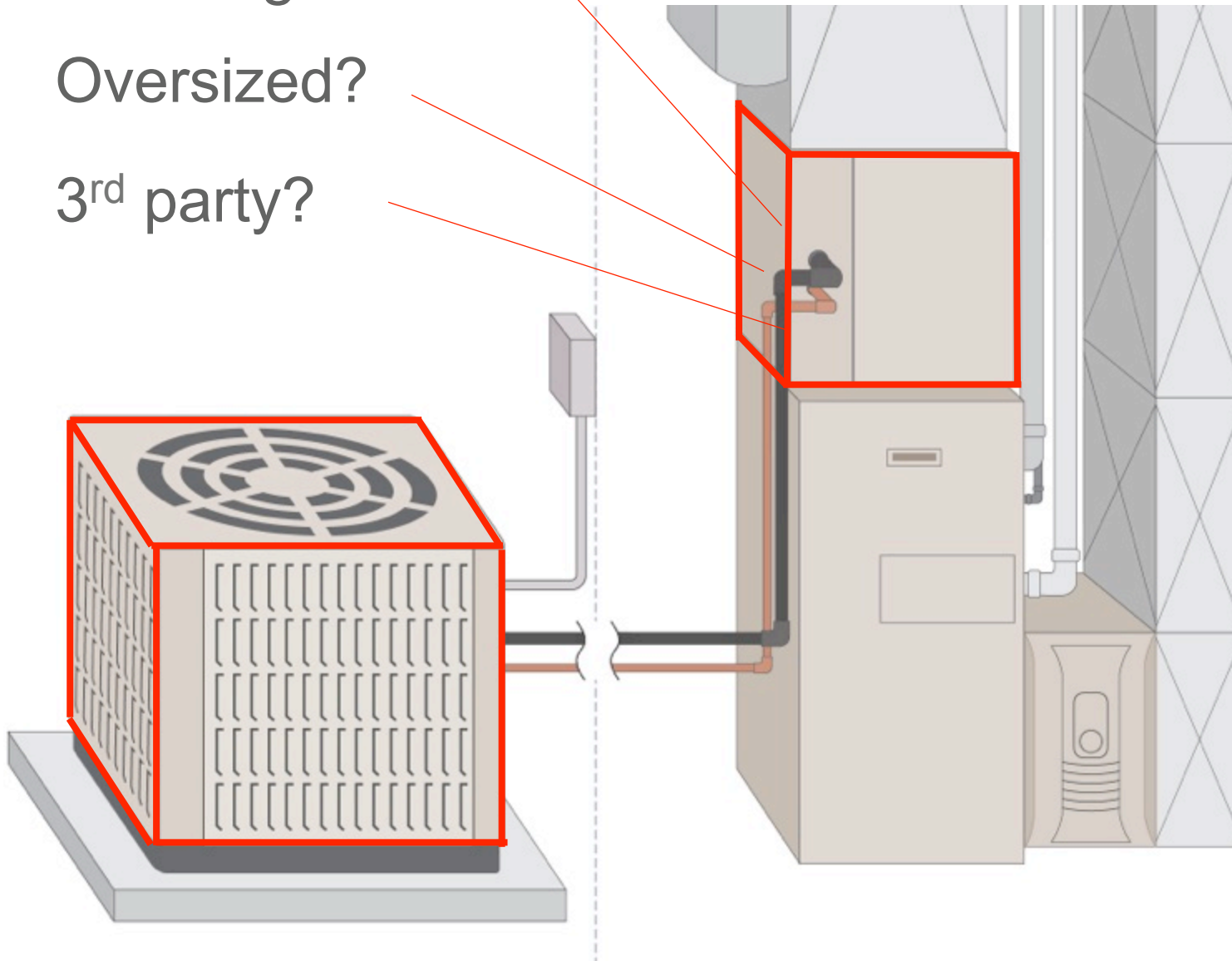
Mini Split



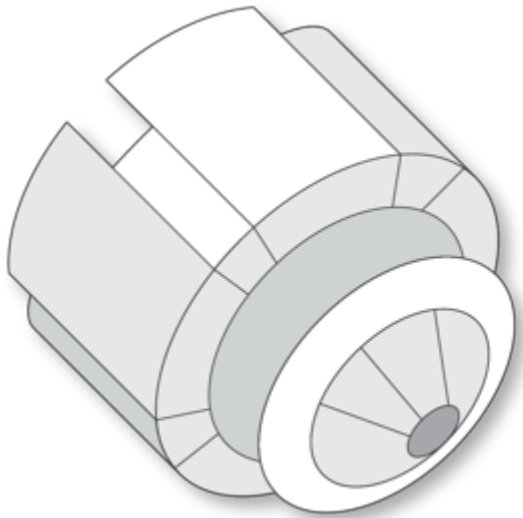
Metering device?

Oversized?

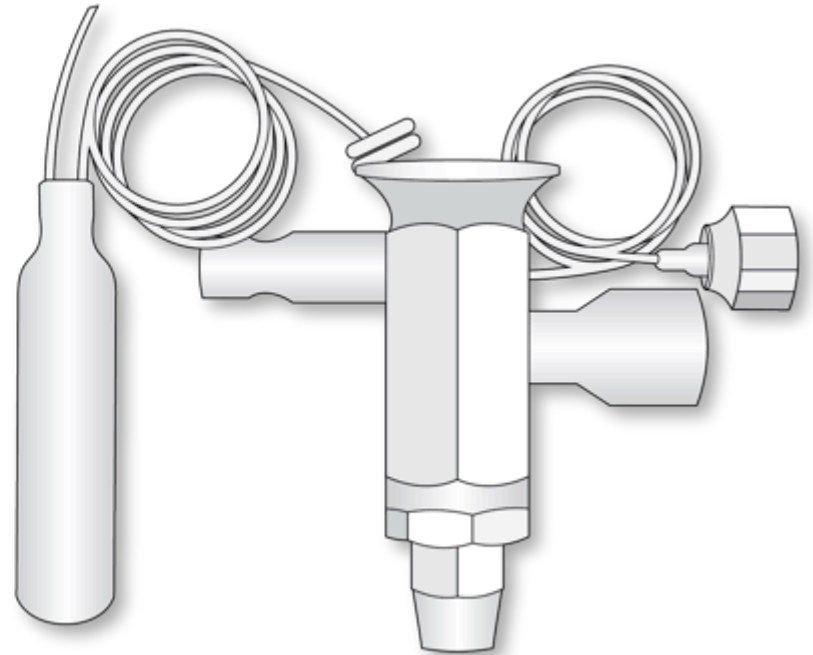
3rd party?



Metering device



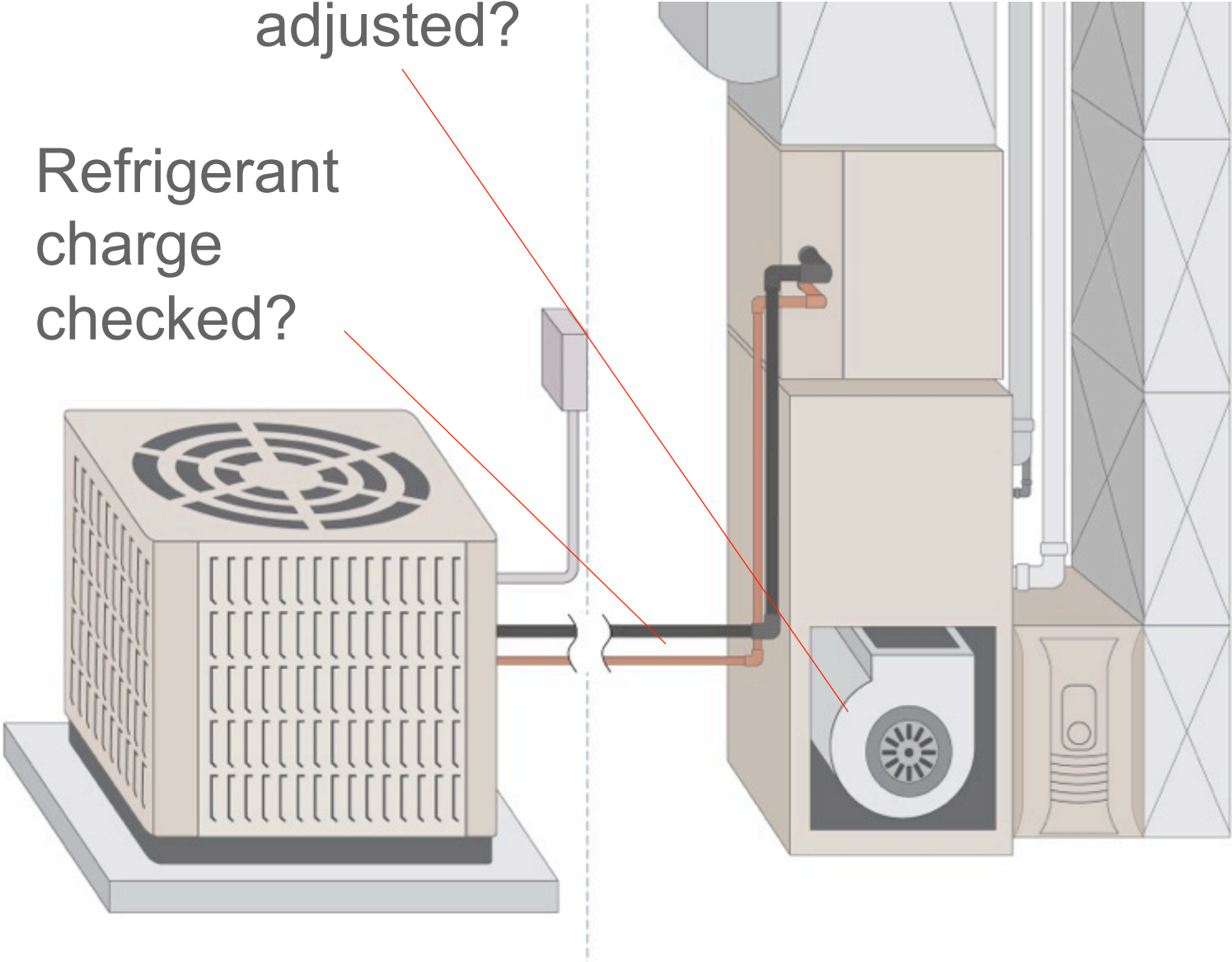
Fixed orifice



Thermostatic
expansion valve (TXV)

Airflow
adjusted?

Refrigerant
charge
checked?



What percent of MN A/C systems have improper refrigerant charge or airflow?

- A. ...10%
- B. ...20%
- C. ...40%
- D. ...60%
- E. ...85%

What percent of MN A/C systems have improper refrigerant charge or airflow?

A. ...10%

B. ...20%

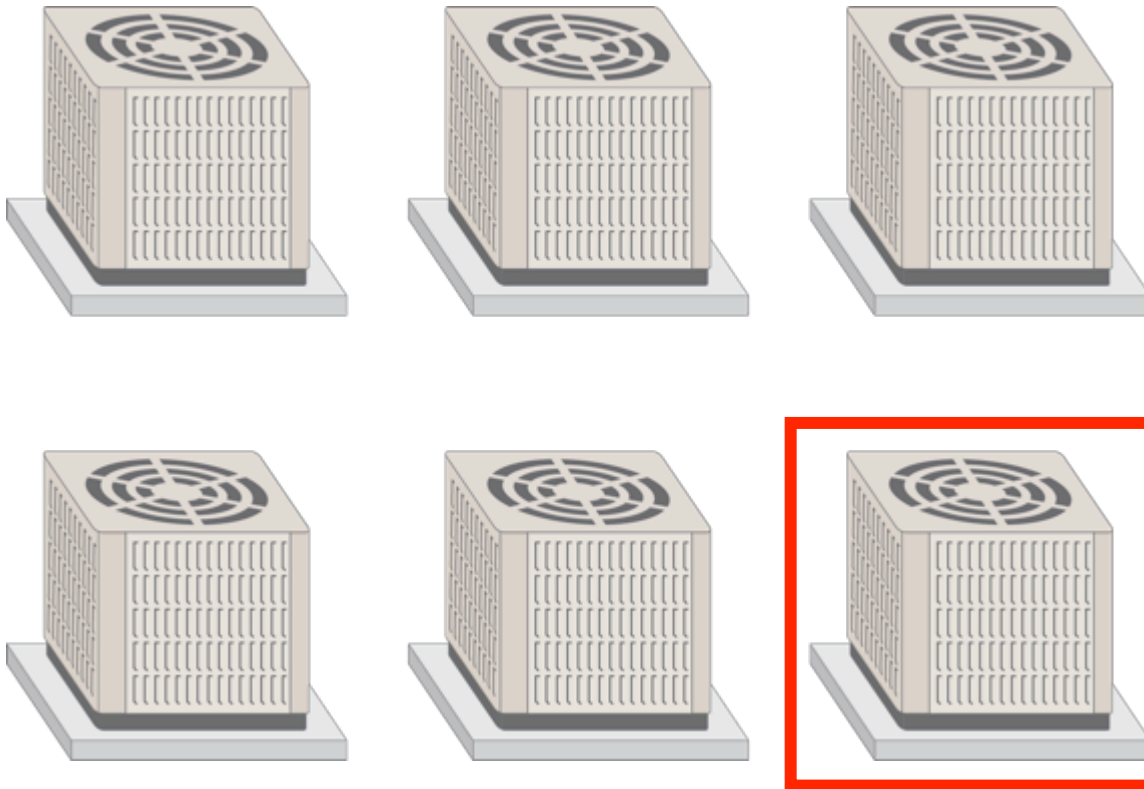
C. ...40%

D. ...60%

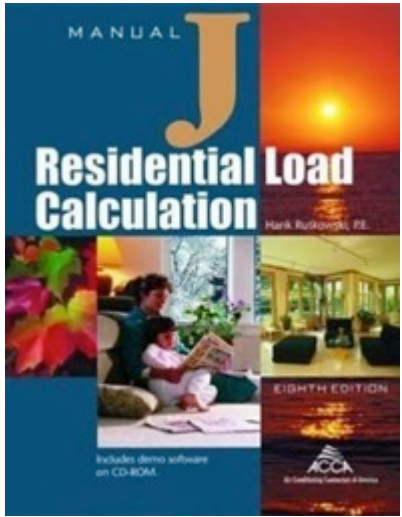
E. ...85%

Typical savings: 10%

One in six can save 25%+



A/C sizing



3 tons



2.5 tons

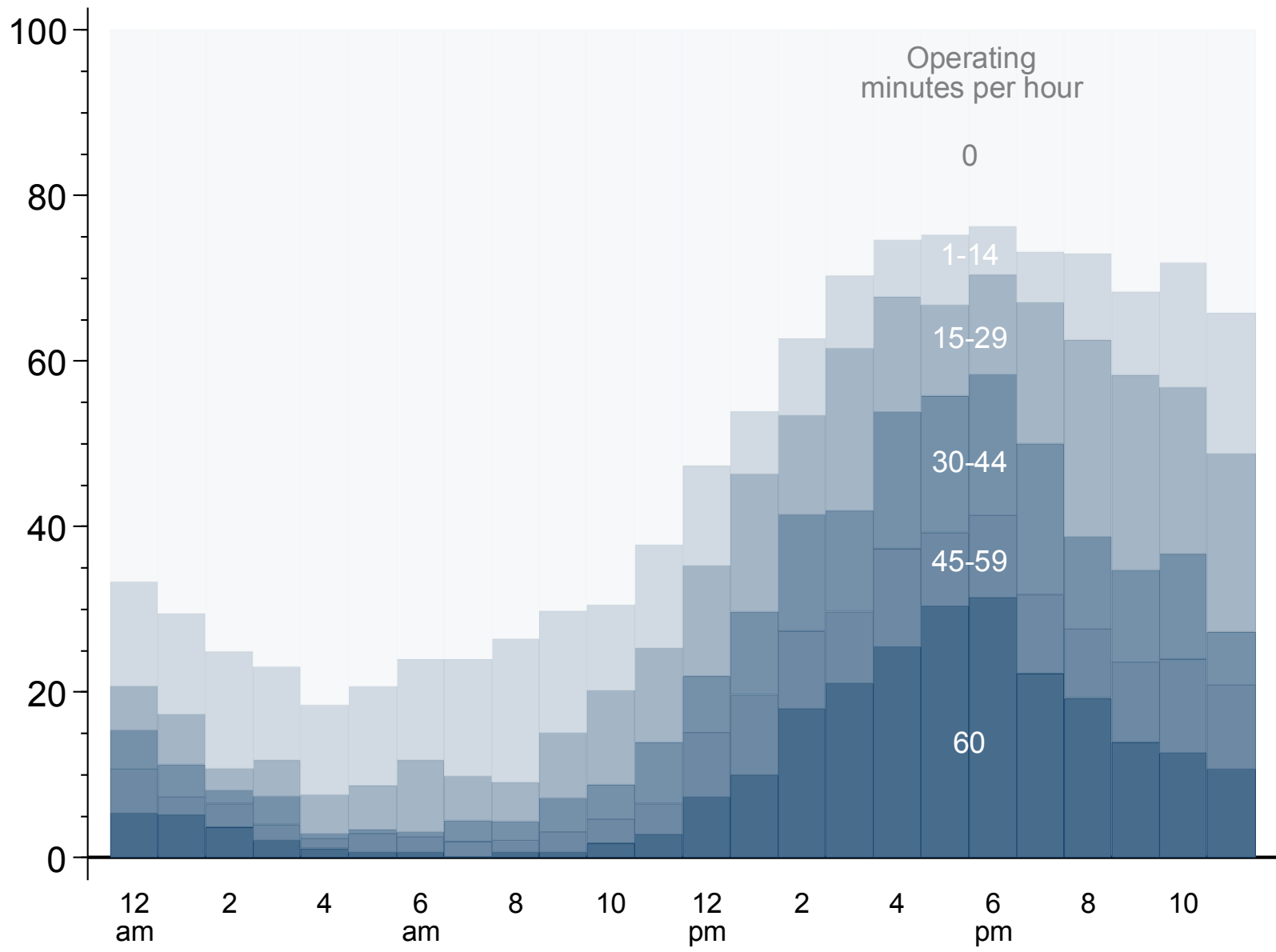


2 tons

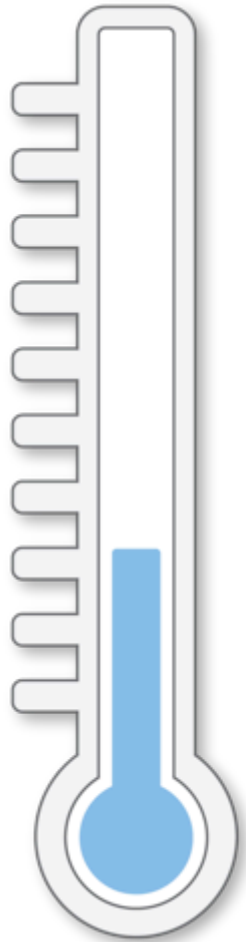


What is a “ton” of cooling?

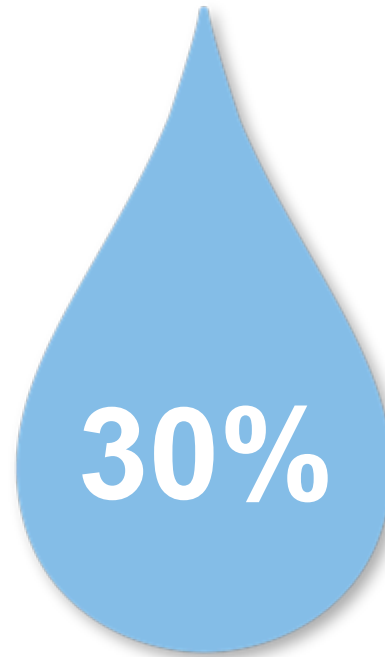




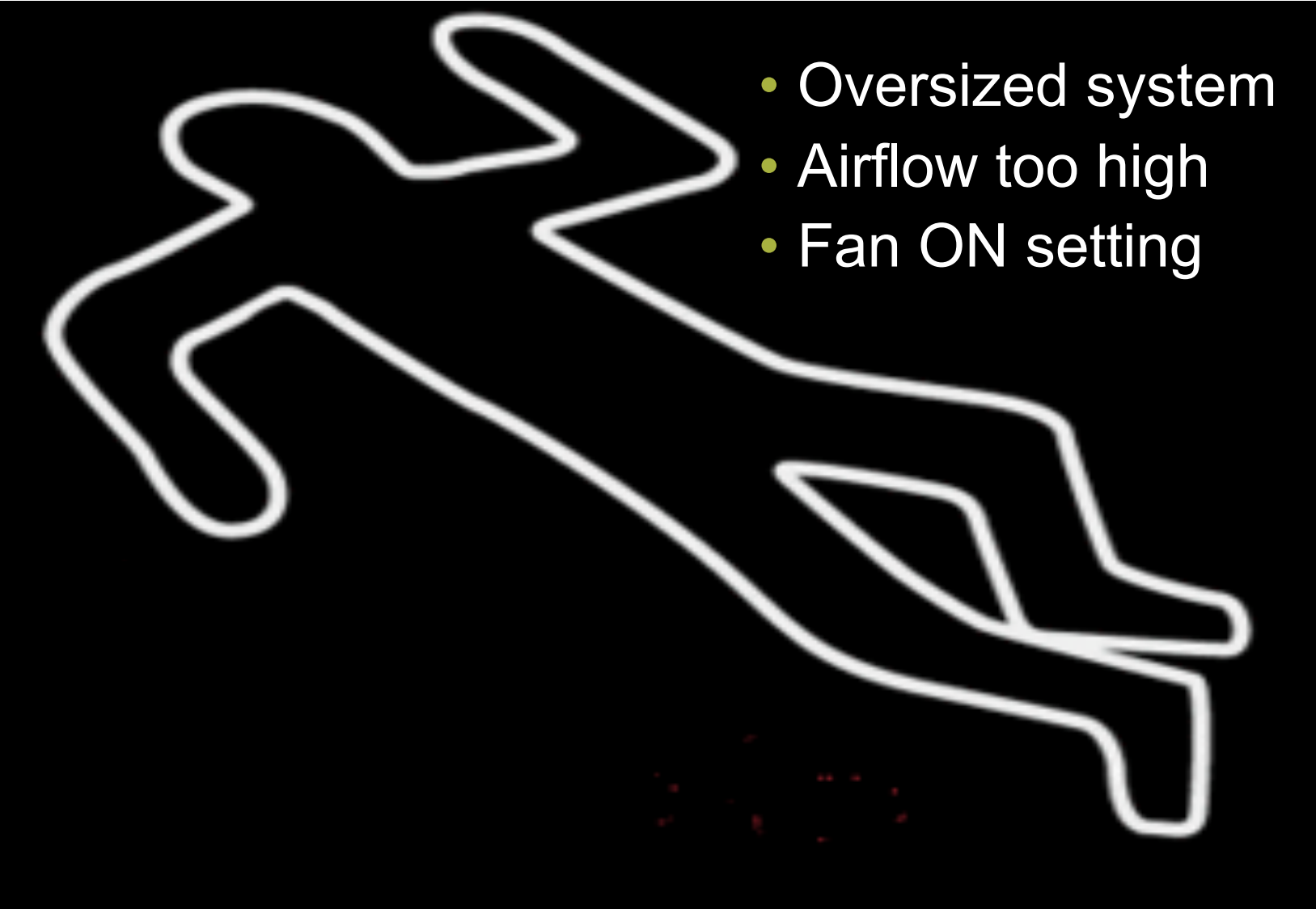
The two jobs of an air conditioner



70%

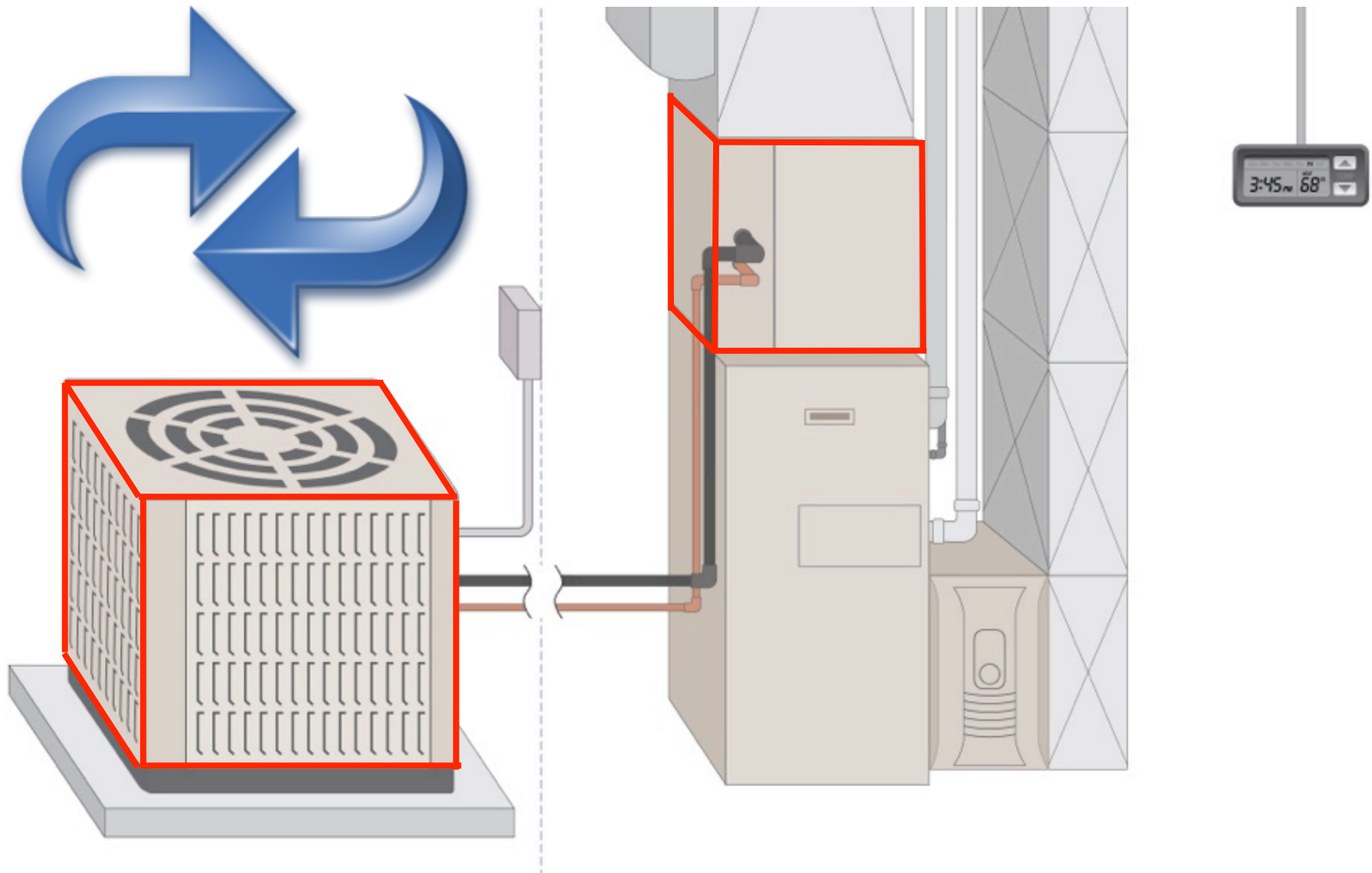


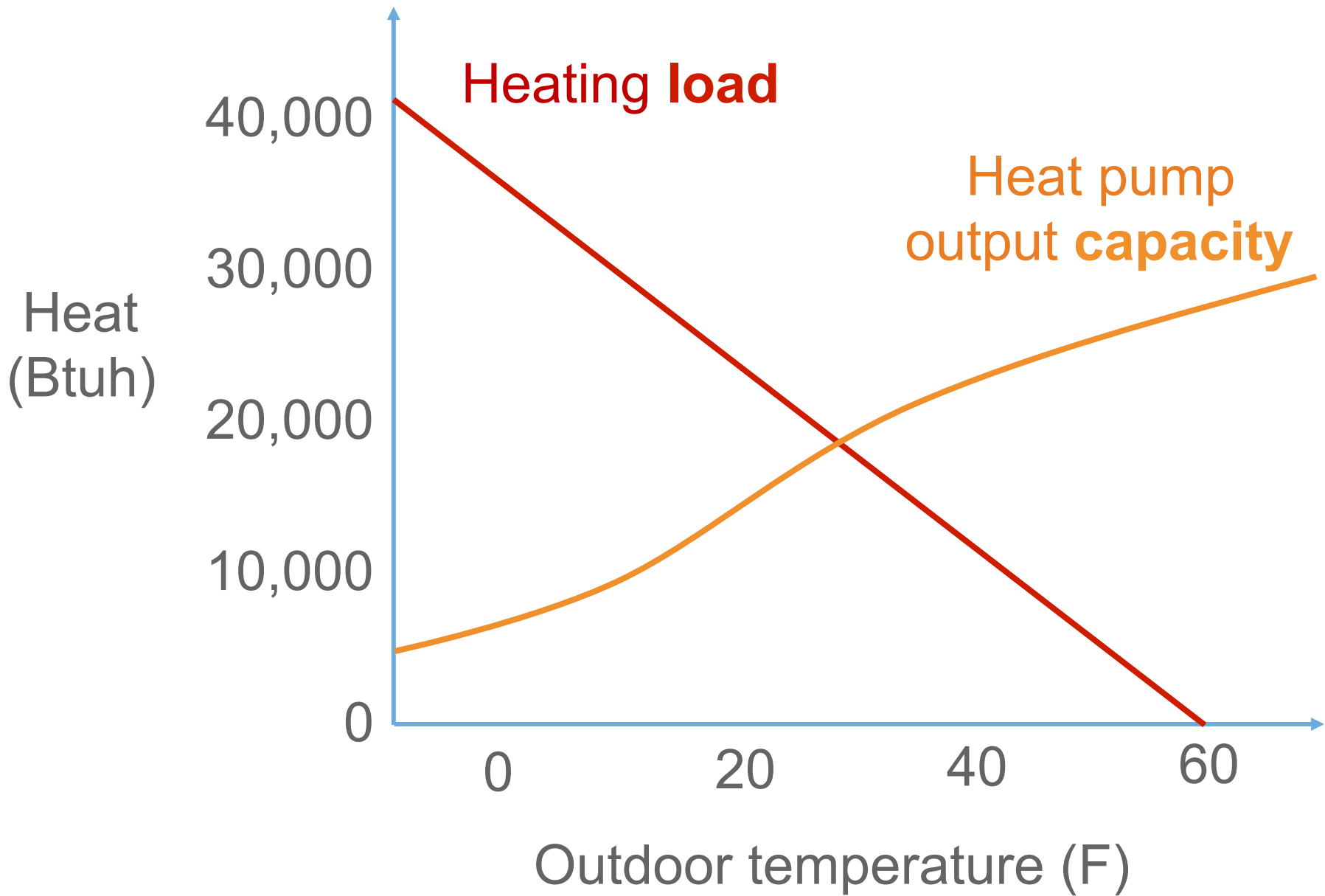
The three dehumidification killers

- 
- Oversized system
 - Airflow too high
 - Fan ON setting

Questions about air conditioners?

What about heat pumps?





| Heating system | | Cost |
|----------------------------------|------|--------|
| Natural gas furnace (condensing) | | \$1.00 |
| Propane furnace (condensing) | | \$1.80 |
| Baseboard electric | | \$4.20 |
| Heat pump | @50F | \$1.20 |
| | @40F | \$1.40 |
| | @30F | \$1.60 |

For:

natural gas at 75 cents per therm

propane at \$1.50 per gallon

electricity at 13 cents per kWh

| Heating system | | Cost |
|----------------------------------|------|---------------------------------|
| Natural gas furnace (condensing) | | \$1.00 |
| Propane furnace (condensing) | | \$1.80 |
| Baseboard electric | | \$4.20 |
| Heat pump | @50F | \$1.20 \$0.55 |
| | @40F | \$1.40 \$0.65 |
| | @30F | \$1.60 \$0.75 |

For:

natural gas at 75 cents per therm

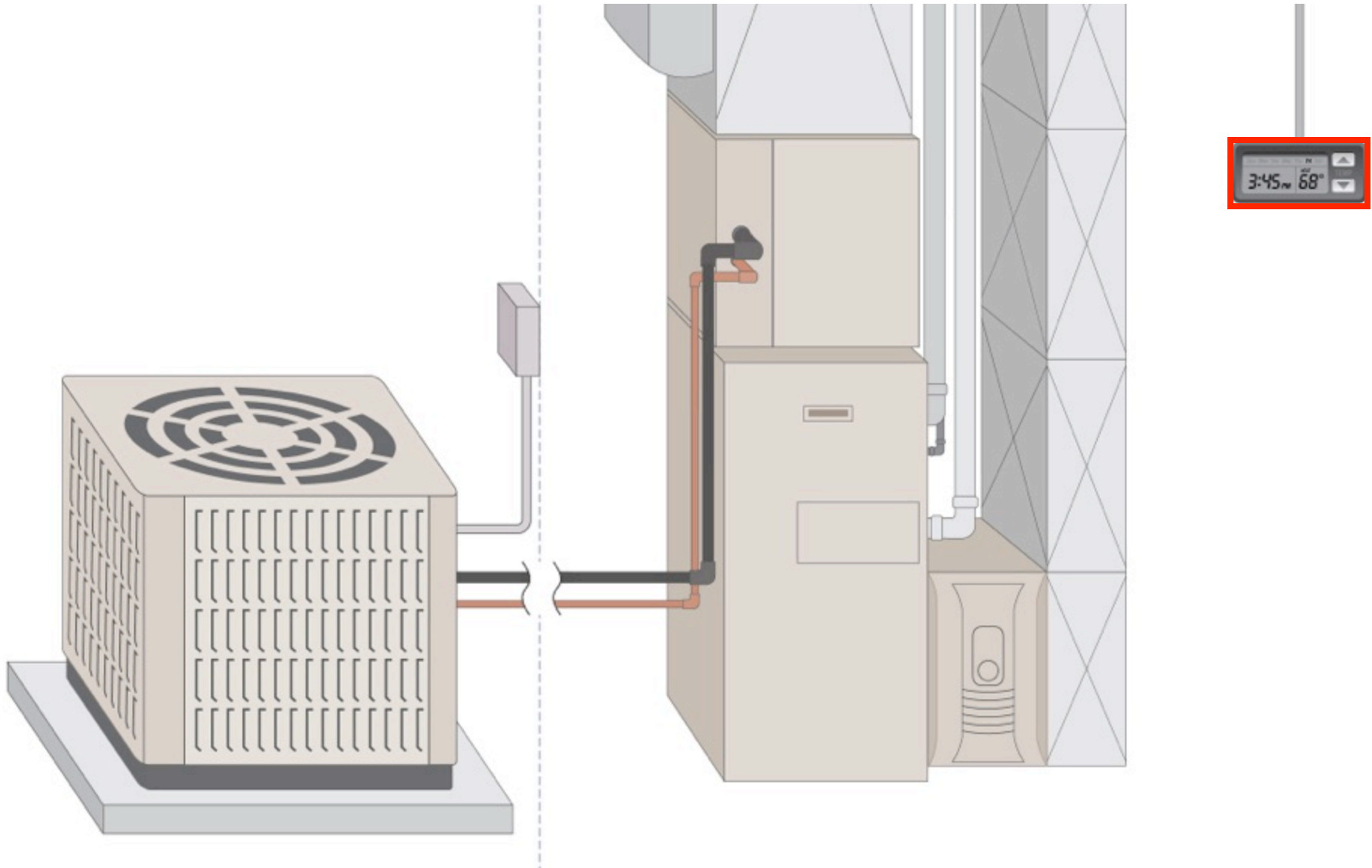
propane at \$1.50 per gallon

electricity at ~~10~~ cents per kWh

6

Questions about heat pumps?

Thermostat

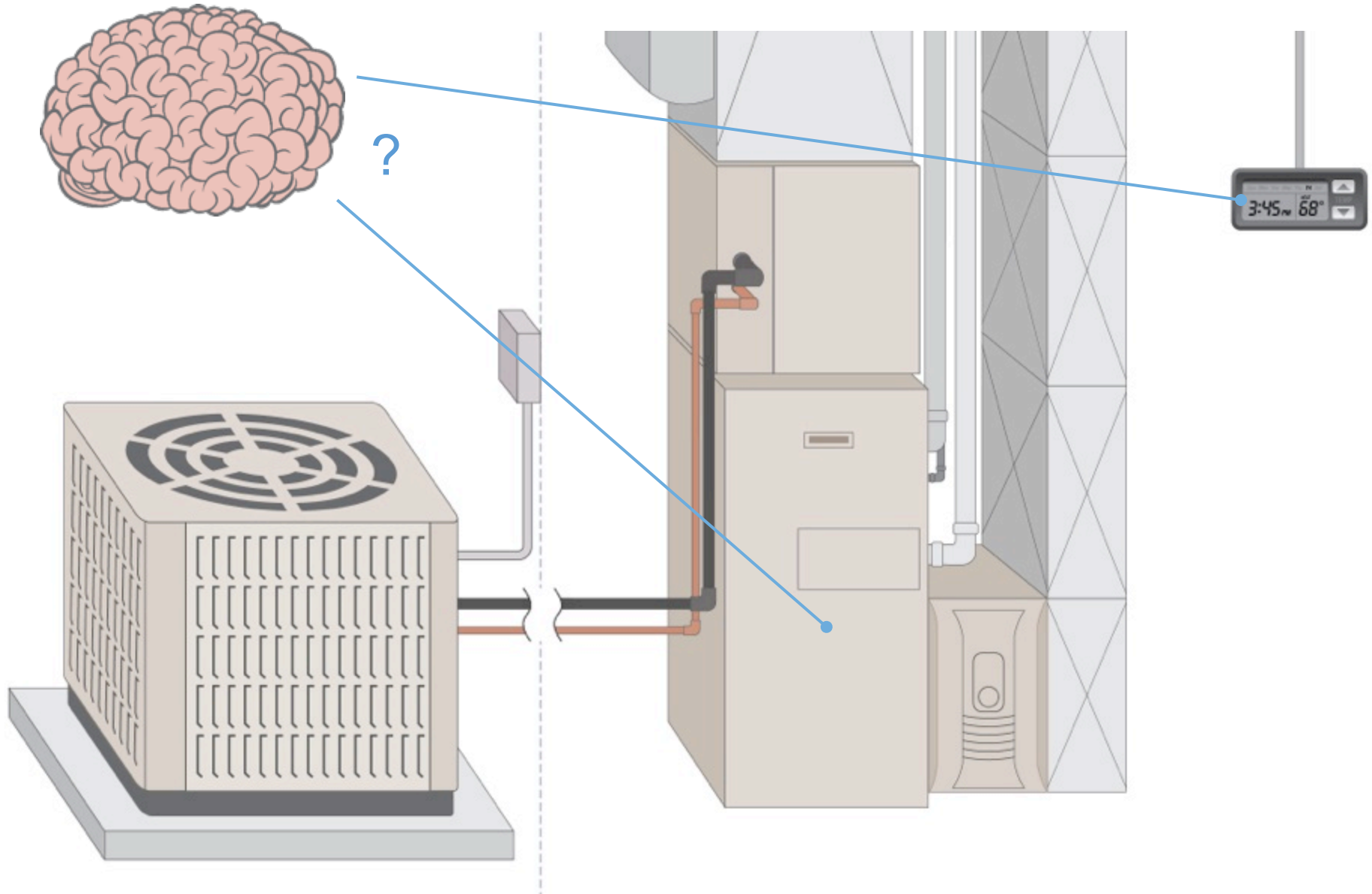


Lots of options

- Programmable?
- Connected?
- “Smart”?
- Proprietary?

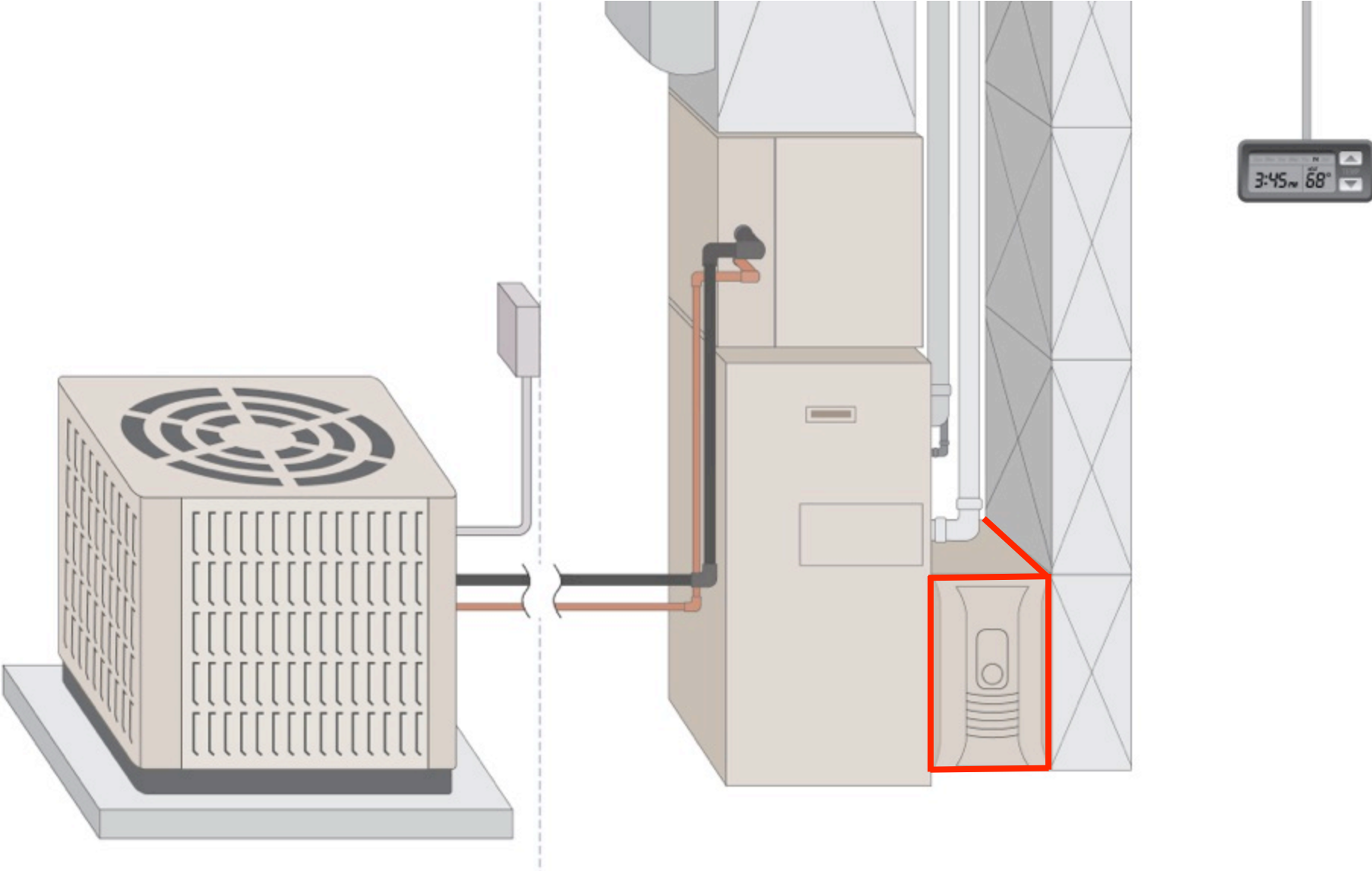


Where's the brain?



Questions about thermostats?

Filter



Options

- 1" disposable
- 4" disposable
- Electrostatic

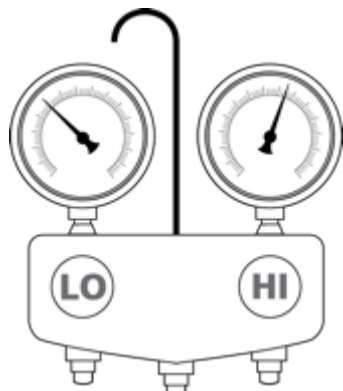
- Filtering efficiency

Minimum Efficiency Reporting Value MERV

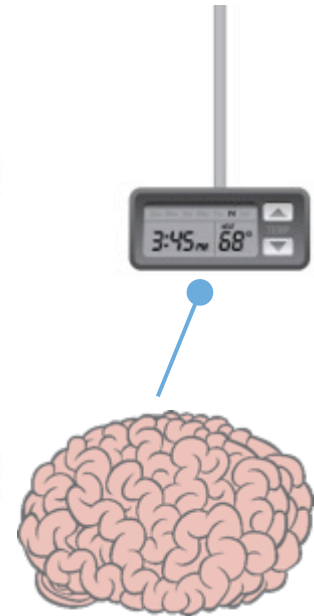
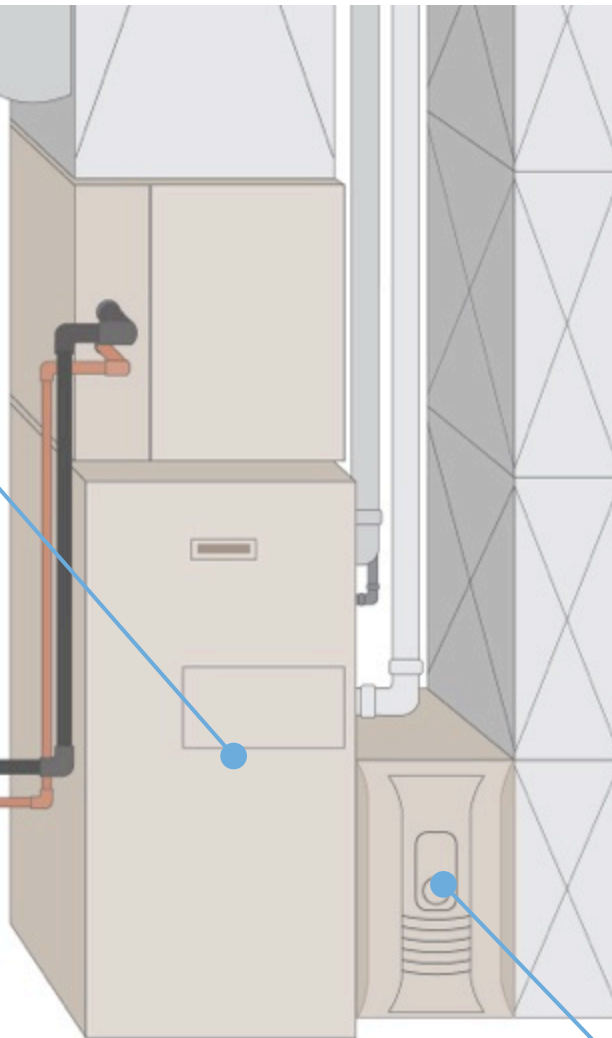
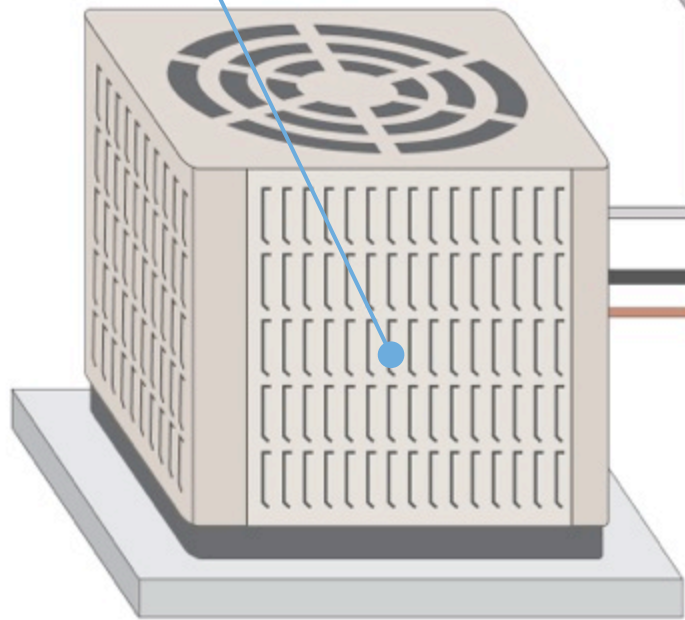
Recipe for disaster



Questions about filters?



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Thank you!

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